

23 June 2010

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SUBJ: TAPER-LOC SYSTEM DRY-GLAZE OR WET-GLAZED  
GRS – GLASS RAIL SYSTEM (SI)

The Taper-Loc System dry-glaze with the GRS Glass Rail System utilizes tapered glass fiber reinforced polycarbonate plates to lock tempered glass in an aluminum extruded base shoe to anchor and support structural glass balustrades which support a variety of top rails and grab rails to construct guards and dividers. The system is intended for interior and exterior weather exposed applications and is suitable for use in all natural environments. The Taper-Loc or wet-glazing with the GRS may be used for residential, commercial and industrial applications. This is an engineered system designed for the loading conditions criteria from the BS 6399-1:1999 "Barriers in and about buildings- Code of practice" and the International Building Code (2006 and 2009).

The base shoe and Taper-Loc® system or wet-glazing with the GRS will meet or exceed the loading indicating in BS6399-1:1996 Table 4 for all occupancy types when secured with appropriate anchorage for the use and substrate. Glass thickness shall be determined from the required loading using the load tables contained herein.

The Taper-Loc with the GRS system will meet or exceed all requirement of BS 6180:1999 "Barriers in and about buildings- Code of practice" as to system performance when properly installed. The Taper-Loc with the GRS system will meet or exceed all requirements of the 2006 and 2009 International Building Codes. Aluminum components are designed in accordance with the 2000 and 2005 Aluminum Design Manuals. Stainless steel components are designed in accordance with SEI/ASCE 8-02 *Specification for the Design of Cold-Formed Stainless Steel Structural Members*.

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Refer to GRS Engineering Report for grab rail mounting.

Signed 06/23/2010

## Typical Installations:

**Surface mounted to steel with anchors @ 300mm o.c.:**

Residential, Commercial and Industrial Applications:

Rail Height 1,100mm above finish floor.

12mm cap screw to steel

<u>Base Shoe</u>	<u>Allowable wind load</u>
Standard/Tapered	Glass strength controls
Heavy	Glass strength controls
Low Profile	Glass strength controls (12, 15mm/16mm glass only)

**8mm anchors to concrete @ 300mm o.c.**

Rail Height 1,100mm above finish floor.

<u>Base Shoe</u>	<u>Allowable wind load</u>
Standard/Tapered	1.40 kN/m <sup>2</sup>
Heavy	1.54 kN/m <sup>2</sup>
Low Profile	1.28 kN/m <sup>2</sup>

**Embedded base shoe:**

All base shoes: Glass strength controls for all cases

**ALLOWABLE LOADS ON GLASS**

Rail Height 1,100mm above finish floor.

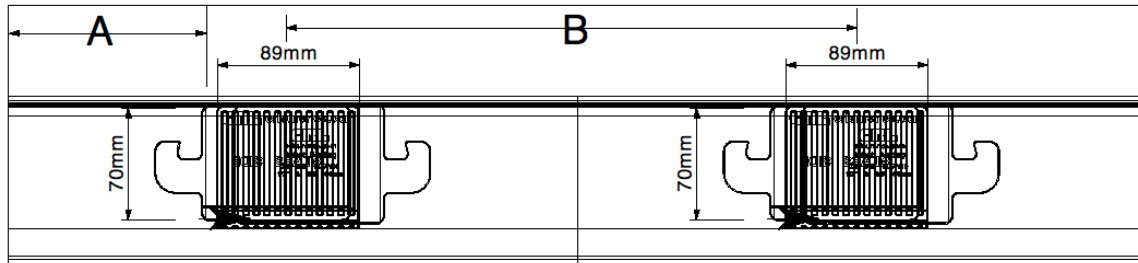
For wet glazed installation:

<u>Glass thickness</u>	<u>Allowable uniform load</u>	<u>Conc load @ top</u>
12mm	1.40 kN/m <sup>2</sup>	0.523kN/m
15mm/16mm	2.725 kN/m <sup>2</sup>	1.02kN/m
19mm	5.0 kN/m <sup>2</sup>	2.07kN/m

For Taper-Loc® alone installation:

<u>Glass thickness</u>	<u>Allowable uniform load</u>	<u>Conc load @ top</u>
12mm	1.40 kN/m <sup>2</sup>	0.523kN/m
15mm/16mm	2.725 kN/m <sup>2</sup>	1.02kN/m
19mm	4.4 kN/m <sup>2</sup>	2.07kN/m

## Taper-Loc® System Typical Installation



For 12mm Fully Tempered Glass and maximum glass height = 1,000mm (1100mm rail height):

Edge Distance:  $51\text{mm} \leq A \leq 219\text{mm}$

Center to center spacing:  $178\text{mm} \leq B \leq 356\text{mm}$  ;

Panel Width/Required quantity of Taper-Loc Plates:

152 to 356mm (6" to 14")	1 TL Plate
356 to 711 mm (14" to 28")	2 TL Plates
711 to 1,067 mm (28" to 42")	3 TL Plates
1,067 to 1,422 mm (42" to 56")	4 TL Plates
1,422 to 1,778 mm (56" to 70")	5 TL Plates
1,778 to 2,134 mm (70" to 84")	6 TL Plates

**Minimum Glass Lite Width =152mm when top rail/guardrail is continuous, welded corners or attached to additional supports at rail ends.**

## NOTES:

- For glass light heights over 1,000  $A_{\max}$  and  $B_{\max}$  shall be reduced proportionally.  
 $A_{\max} = 219 * (1,000/h)\text{mm}$ ;  
 $B_{\max} = 356 * (1,000/h)\text{mm}$ ;
- For glass light heights under 1,000mm  $A_{\max}$  and  $B_{\max}$  shall not be increased.
- $A_{\min}$  and  $B_{\min}$  are for ease of installation and can be further reduced as long as proper installation is achieved.
- For glass thicknesses greater than 12mm  $A_{\max}$  and  $B_{\max}$  may be increased as follows:  
 15mm/16mm Glass  
 Edge Distance:  $51\text{mm} \leq A \leq 343\text{mm}$ ;  
 Center to center spacing:  $178\text{mm} \leq B \leq 533\text{mm}$ ;  
 19mm Glass  
 Edge Distance:  $51\text{mm} \leq A \leq 483\text{mm}$ ;  
 Center to center spacing:  $178\text{mm} \leq b \leq 787\text{mm}$

**$A_{\max}$  and  $B_{\max}$  shall be reduced proportionally for heights over 1,000mm or loads greater than indicated for 12mm glass.**

**LOAD CASES:**

Dead load =

0.026kN/m top rail

0.125kN/m base shoe

Loading:

Horizontal load to base shoe

$1.5\text{kN/m}^2 \cdot H$  or  $W \cdot H$

Balustrade moments

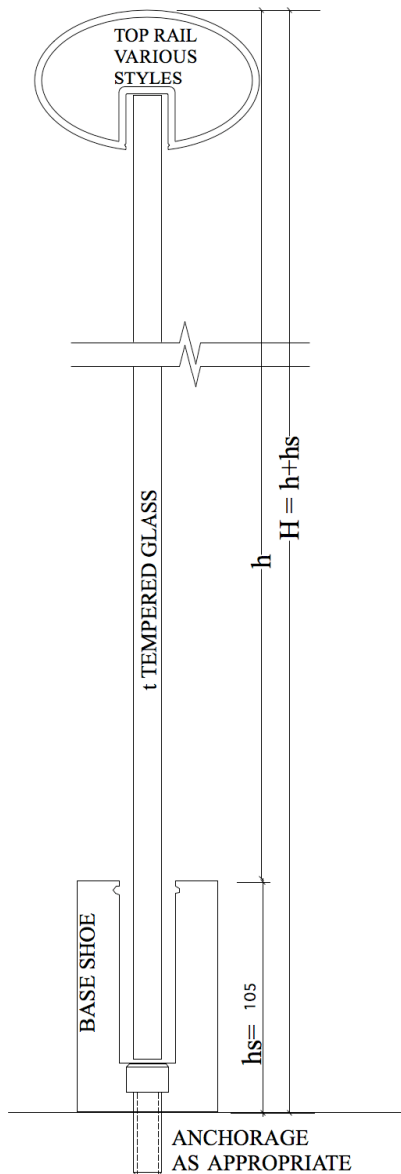
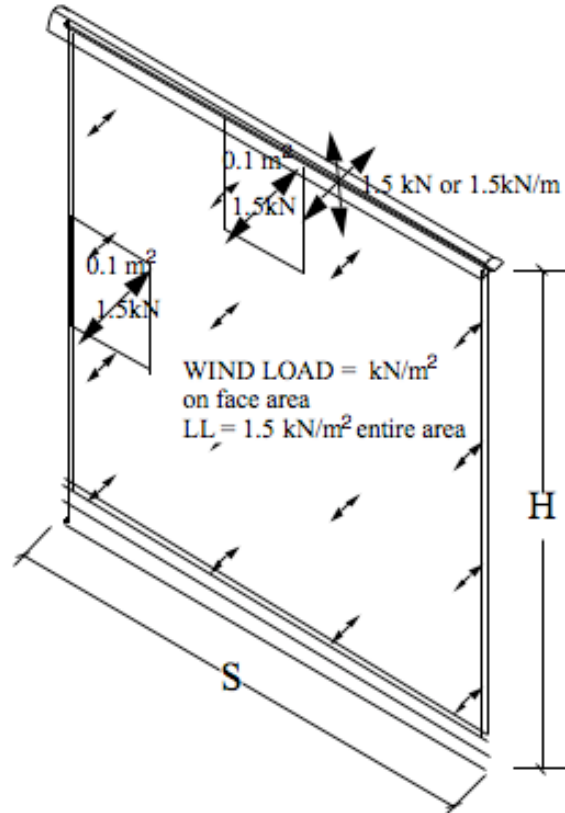
$M_i = 1.5\text{kN/m}^2 \cdot H^2 / 2$  or

$M_w = 1.5\text{kN/m}^2 \cdot H^2 / 2$

For top rail loads:

$M_c = 1.5\text{kN} \cdot H$

$M_u = 1.5\text{kN/m} \cdot H$



Three options for glass thickness:

12mm glass, weight =  $0.309\text{ kN/m}^2$

15mm/16mm glass, weight =  $0.385\text{ kN/m}^2$

19.0mm glass, weight =  $0.448\text{ kN/m}^2$

For 1,100mm rail height:

$h = 1100 - 100\text{mm} = 1,000\text{mm}$

The IBC design loading conditions are:

On Top Rail:

Concentrated load =  $0.89\text{ kN}$  (200 lbs) any direction, any location

Uniform load =  $0.73\text{ kN/m}$  (50 plf), any direction perpendicular to rail

On In-fill Panels:

Concentrated load =  $0.22\text{ kN}$  on  $0.093\text{ m}^2$  (50# on one sf).

Distributed load =  $1.2\text{ kN/m}^2$  (25 psf) on area of in-fill, including spaces

Wind load = As stated for the application and components

Refer to IBC Section 1607.7.1

## GLASS BALUSTRADE GUARD RAIL

## GLASS STRENGTH

All glass is toughened (fully tempered glass) conforming to the specifications of BS6206:1981, BS6262 and BS6262-4. The minimum Modulus of Rupture  $F_r$  is 165.5 MPa.

$$E = 71,705 \text{ MPa} = 71.7 \times 10^9 \text{ N/m}^2$$

Allowable glass bending stress:  $165.5 \text{ MPa}/4 = 41.375 \text{ MPa}$ . – Tension stress calculated.

Bending strength of glass for the given thickness:

$$I = \frac{1,000\text{mm} \cdot (t)^2}{12} = 83.3 \cdot (t)^2 \text{ mm}^3/\text{m}$$

$$S = \frac{1,000\text{mm} \cdot (t)^2}{6} = 166.7 \cdot (t)^2 \text{ mm}^3/\text{m}$$

For 12mm glass

$$I = 83.3 \cdot (12)^3 = 144,000 \text{ mm}^4/\text{m}$$

$$S = 166.7 \cdot (12)^2 = 24,000 \text{ mm}^3/\text{m}$$

$$M_{\text{allowable}} = 41.375 \text{ MPa} \cdot 24,000 \text{ mm}^3/\text{m} / 10^9 = 0.993 \text{ kN-m/m}$$

For cantilevered elements basic beam theory for cantilevered beams is used.

$$M_w = W \cdot L^2 / 2 \text{ for uniform load } W \text{ and span } L \text{ or}$$

$$M_p = P \cdot L \text{ for concentrated load } P \text{ and span } L,$$

Need to check deflection:

$$\Delta = w l^4 / (8EI) \text{ or}$$

$$\Delta = P l^3 / (3EI)$$

$$\Delta_{\text{all}} \leq 1/65 = 1100/65 = 16.9 \text{ mm} \leq 25 \text{ mm}$$

Determine maximum allowable loads on 12 mm glass:

From stress:

$$\text{Top load} = 993 \text{ Nm}/1.0 \text{ m} = 993 \text{ N per meter}$$

$$\text{Glass uniform load} = 2 \cdot 993 \text{ Nm}/1.0^2 = 1.986 \text{ kN/m}^2$$

From deflections

$$\text{Infill } \omega = 16.9 \text{ mm} \cdot (8 \cdot 71.7 \times 10^9 \text{ N/m}^2 \cdot 144,000 \text{ mm}^4/\text{m}) / (1000 \text{ mm})^4 = 1.40 \text{ kN/m}^2$$

$$\text{Top } P = 16.9 \text{ mm} \cdot (3 \cdot 71.7 \times 10^9 \text{ N/m}^2 \cdot 144,000 \text{ mm}^4/\text{m}) / (1000 \text{ mm})^3 = 523 \text{ N/m}$$

For 15mm or 16mm glass (references to 15 mm thickness includes 16mm)

$$I = 83.3 \cdot (15)^3 = 281,137 \text{ mm}^4/\text{m}$$

$$S = 166.7 \cdot (15)^2 = 37,507 \text{ mm}^3/\text{m}$$

$$M_{\text{all}} = 41.37 \text{ MPa} \cdot 37,507 / 10^8 = 1.55 \text{ kNm/m}$$

From stress:

$$\text{Top load} = 1.55 \text{ kNm} / 1.0 \text{ m} = 1.55 \text{ kN per meter}$$

$$\text{Glass uniform load} = 2 \cdot 1.55 \text{ kNm} / 1.0^2 = 3.10 \text{ kN/m}^2$$

From deflections

$$\text{Infill } \omega = 16.9 \text{ mm} \cdot (8 \cdot 71.7 \times 10^9 \text{ N/m}^2 \cdot 281,137 \text{ mm}^4/\text{m}) / (1000 \text{ mm})^4 = 2.725 \text{ kN/m}^2$$

$$\text{Top } P = 16.9 \text{ mm} \cdot (3 \cdot 71.7 \times 10^9 \text{ N/m}^2 \cdot 281,137 \text{ mm}^4/\text{m}) / (1000 \text{ mm})^3 = 1.02 \text{ kN/m}$$

For 19mm glass

$$I = 83.3 \cdot (19)^3 = 571,355 \text{ mm}^4/\text{m}$$

$$S = 166.7 \cdot 19^2 = 60,178.7 \text{ mm}^3/\text{m}$$

$$M_{\text{all}} = 41.37 \text{ MPa} \cdot 60,178.7 / 10^8 = 2.5 \text{ kNm/m}$$

Maximum uniform load based on glass strength

$$w = (2.5 \text{ kNm/m} \cdot 2) / (h^2)$$

$$h = \sqrt{(2.5 \text{ kNm/m} \cdot 2 / w)}$$

1,100mm (42") guardrail height: glass lite height = 1,000 mm

Calculate maximum uniform or wind load on glass:

$$w = (2.5 \text{ kNm/m} \cdot 2) / (0.1^2) = 5.0 \text{ kPa}$$

$$\text{Top load} = 2.5 \text{ kNm} / 1.0 \text{ m} = 2.5 \text{ kN per meter}$$

From deflections

$$\text{Infill } \omega = 16.9 \text{ mm} \cdot (8 \cdot 71.7 \times 10^9 \text{ N/m}^2 \cdot 571,355 \text{ mm}^4/\text{m}) / (1000 \text{ mm})^4 = 5.538 \text{ kN/m}^2$$

$$\text{Top } P = 16.9 \text{ mm} \cdot (3 \cdot 71.7 \times 10^9 \text{ N/m}^2 \cdot 571,355 \text{ mm}^4/\text{m}) / (1000 \text{ mm})^3 = 2.073 \text{ kN/m}$$

**NOTE: FOR THE TAPER-LOC SYSTEM INSTALLED WITHOUT WET GLAZING  
GLASS LOADS TYPICALLY DO NOT NEED TO BE ADJUSTED FOR STRESS  
CONCENTRATIONS AS SHOWN LATER IN THIS REPORT.**

No adjustment to allowable loads calculated from deflection limits are required.



## DRY-GLAZE TAPER-LOC SYSTEM

Glass is clamped inside the aluminum base shoe by the Taper-Loc Shoe Setting Plate (L shaped piece on the back side) and two Taper-Loc Shim Plates (front side). The glass is locked in place by the compressive forces created by the Taper-Loc shim plates being compressed together by the installation tool. Use of the calibrated installation tool assures that the proper compressive forces are developed. Until the shim plates are fully installed the glass may be moved within the base shoe for adjustment.

Glass may be extracted by reversing the installation tool to extract tapers.

The Taper-Loc setting plate is bonded to the glass by adhesive tape to hold it in place during installation and to improve glass retention in the base shoe.

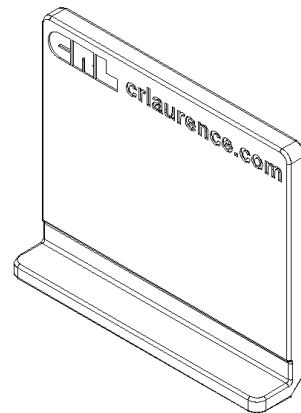
Surface area of the setting plate adhered to the glass:

$$A = 50.8\text{mm} \times 88.9 = 4,516\text{mm}^2 \quad (2'' \times 3.5'' = 7\text{ in}^2)$$

$$\text{adhesive shear strength} \geq 551.6\text{ kN/m}^2 \quad (80\text{ psi})$$

3M™ VHB Tape

$$Z = 4,516\text{mm}^2 \times 551.6\text{ kN/m}^2 = 2.49\text{kN} \quad (7\text{ in}^2 \times 80 = 560\# \text{ min})$$



The setting plate locks into place in the base shoe by friction created by the compression generated when the shim plates are locked into place.

Installation force:

$$T_{\text{des}} = 28.2\text{Nm} \quad (250\#''\text{'}) \text{ design}$$

installation torque

$$T_{\text{max}} = 33.9\text{Nm} \quad (300\#''\text{'}) \text{ maximum}$$

installation torque

Compressive force generated by the

installation torque:

$$C = (0.2 \times 28.2\text{Nm} / 25.4\text{mm}) / \sin(1.76^\circ)$$

$$C = 7.24\text{kN} \quad (1,628\#)$$

Frictional force of shims and setting plate against aluminum base shoe:

coefficient of friction,  $\mu = 0.65$

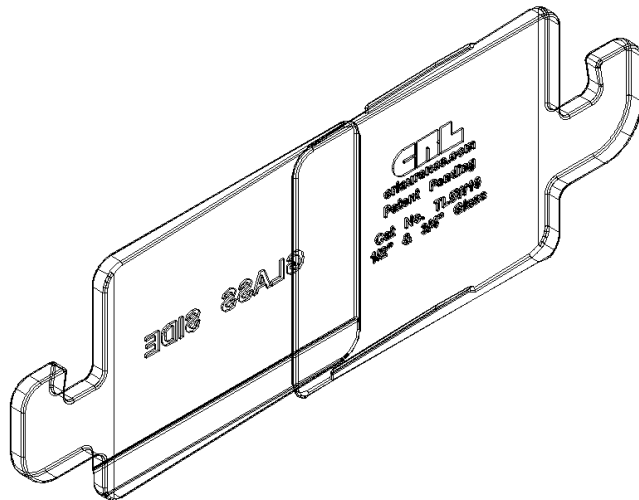
$$f = 2 \times (7.24\text{kN} \times 0.65) = 9.37\text{kN}$$

Frictional force of shims against glass:

$\mu = 0.20$

$$f = 7.24\text{kN} \times 0.20 = 1.448\text{kN}$$

Resistance to glass pull out:



$$U = 2.491\text{kN} + 1.448\text{kN} = 3.939\text{kN}$$

Safety factor for 890N (200#) pullout resistance =  $3.939/0.89 = 4.43$

Minimum recommended installation torque:

$$4/4.43 * 28.2\text{Nm} = 25.5\text{Nm}$$

Extraction force required to remove tapers after installation at design torque:

$$T = 28.2\text{Nm} * (0.7/0.2) = 98.7\text{Nm} \quad (875\#")$$

Glass anchorage against overturning:

Determine reactions of Taper-Loc plates on the glass:

Assuming elastic bearing on the nylon parts the reactions will have centroids at approximately  $1/6 * 64.77\text{mm}$  (2.55") from the upper and lower edges of the bearing surfaces:

$$R_{Cu} @ 1/6 * 64.77\text{mm} = 10.8\text{mm} \quad (0.425")$$

From  $\sum M$  about  $R_{Cu} = 0$

$$0 = M + V * (10.8\text{mm} * 12.7\text{mm}) - R_{Cb} * 43.2\text{mm}$$

Where  $M = V * 965.2\text{mm}$

substitute and simplify:

$$0 = V * 988.7\text{mm} - R_{Cb} * 43.2\text{mm}$$

Solving for -  $R_{Cb}$

$$R_{Cb} = V * 988.7 / 43.2 = 22.9V$$

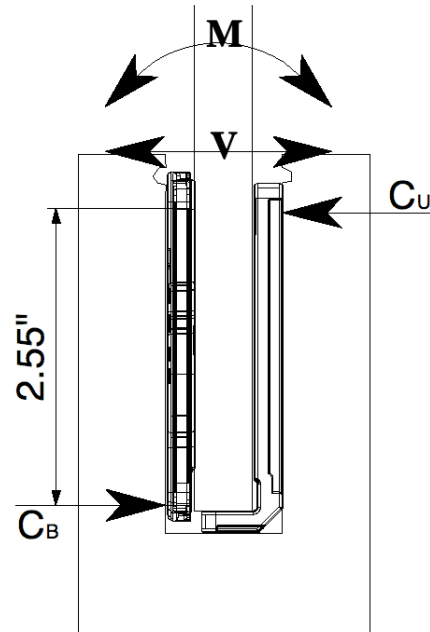
For  $C_B = 20.68\text{MPa}$  (3,000 psi):

$$R_{Cb} = 88.9\text{mm} * (64.77\text{mm} / 2) * 20.68\text{MPa} / 2 = 29.78\text{kN}$$

$$V_a = 29.78\text{kN} / 22.9 = 1.3\text{kN}$$

$$M_a = R_{Cb} * (2/3 * 64.77\text{mm}) = 1.286\text{kNm}$$

$$R_{Cb} = R_{Cb} + V = 29.78\text{kN} + 1.299\text{kN} = 31.075\text{kN}$$



At maximum allowable moment determine bending in base shoe legs:

$$M_s = C * (0.188 + 2.55"/2) + R_{Cb} * (0.188 + 2.55 - 0.425) =$$

$$M_s = 1,954 * (1.463) + 6,986 * (2.313) = 2.149\text{kNm}$$

Base shoe tributary length of leg that resists bending from load:

$L = 88.9\text{mm} + 8 * 12.7\text{mm} + 2 * (82.55\text{mm}) = 355.6\text{mm}$  ( $3.5" + 8 * 0.5" + 2 * (3.25") = 14"$ ), This is the maximum allowable spacing of the Taper-Loc system so represents the maximum loading condition.

$$\text{Strength of leg } 355.6\text{mm} (14") \text{ length} = 5.212\text{kNm} * 0.3556 = 1.854\text{kNm}$$

Adjustment to allowable load based on base shoe strength:

$$M_a = 1.854\text{kNm} / 2.149\text{kNm} * 1.286\text{kNm} = 1.109\text{kNm} \quad (16,406 / 19,017 * 11,380 = 9,818\#")$$

Allowable Moment per lineal meter of glass rail:

$$M_a = 1.109\text{kNm} * (1000\text{mm} / 355.6\text{mm}) = 3.118\text{kNm} \quad (8,415\#"/\text{ft})$$

**GLASS STRESS ADJUSTMENTS FOR THE TAPER-LOC SYSTEM**

The Taper-Loc System provides a concentrated support:

Stress concentration factor on glass based on maximum 356mm glass width to each Taper-Loc set.

Moment concentration factor

$$C_M = [1 + (1 - a/b)^2 (1 - c/b)^3 (1 - t/b)^{1/3}]^{1/2}$$

$a = 63.5\text{mm}$  (2.75") (bottom of glass to top of bearing)

$b =$  center to center spacing of supports or width of glass.

$c =$  length of bearing

glass thickness will have less than 1% change in the stress concentration so can be ignored for the three glass thicknesses.

$$C_M = [1 + (1 - 63.5/356)^2 (1 - 89/356)^3 (1 - 12.7/356)^{1/3}]^{1/2} = 1.13$$

$b/h = 356\text{mm}/817\text{mm} = 0.4375 < 1$  based on maximum spacing of 356mm and minimum glass height of 817mm (920mm rail)

$$C_{M'} = 1 + (C_M - 1) * (b/h)^3 = 1.01$$

Since adjustment is typically under 1% it can be ignored when glass height exceeds 21" since  $C_{M'} < 1.04$

$$F_b = 41.37\text{MPa} (6,000\text{psi})$$

Shear concentration factor:

$$C_V = 356/89 * (2 - 89/356) = 7.0$$

$$F_{V_a} = 20.68\text{MPa} (3,000\text{psi}) \text{ maximum allowable shear stress}$$

Since shear load in all scenarios is under 10% of allowable it can be ignored in determining allowable bending since it has less than 1% impact on allowable bending loads or rail heights.

Maximum edge distance for edge of glass to centerline of Taper-Loc plates:

$$e_{des} = 356/2 = 178\text{mm} \text{ for design conditions (no reduction in allowable),}$$

$$C_M = 1.0436 \text{ for cantilever moment.}$$

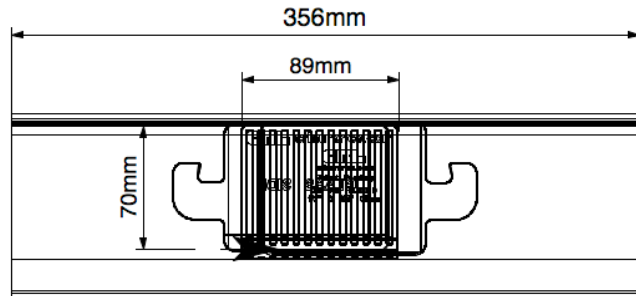
$$e_{max} = e + e_{des}/2: \text{ limited by added glass stress from cantilever bending of glass}$$

$$e_{max} = e + e_{des}/2:$$

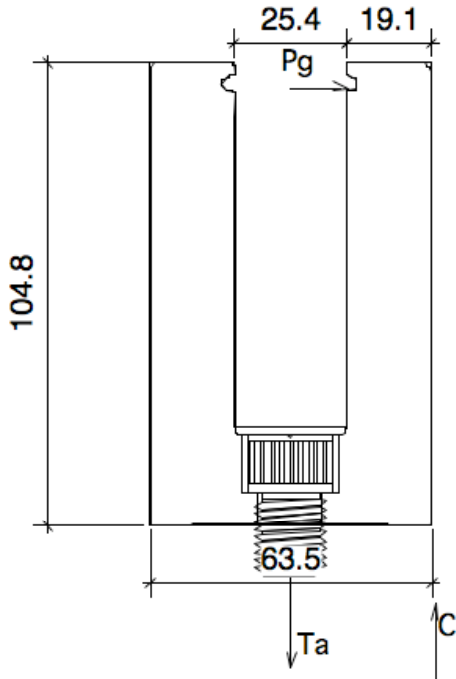
$$(e) + 1.25 * (1100\text{mm})/2 = 1.0436 * 993\text{N-m}/1200\text{N/m}^2 = 863.6\text{mm}:$$

$$\text{solve for } e: e = 177$$

$$e_{max} = 178\text{mm}/2 + 177 = 266\text{mm} \text{ (to CL of Taper-Loc plates)}$$



**B5S 105mm x 63mm GLASS BALUSTRADE BASE SHOE**



6063-T52 Aluminum extrusion

Fully tempered glass glazed in place with Taper-Loc® dry-glaze.

Shoe strength – Vertical legs:

Glass reaction by bearing on legs to form couple.

Allowable moment on legs:

$$M_a = S_1 * F_t \text{ or } F_c$$

$$F_t = F_c = 86.18 \text{ MPa (12.5 ksi) (ADM Table 2-23, Sec 3.4.4)}$$

$$S_1 = 1000 \text{ mm} * (19.1 \text{ mm})^2 / 6 = 60,802 \text{ mm}^3$$

$$M_a = 60,802 \text{ mm}^3 * 86.18 \text{ MPa} = 5.24 \text{ kNm/m}$$

Leg shear strength @ bottom

$$t_{\min} = 19.1 \text{ mm (0.75")}$$

$$F_v = 58.6 \text{ MPa 8.5 ksi (ADM Table 2-24, Sec 3.4.20)}$$

$$V_{\text{all}} = 19.1 \text{ mm} * 1,000 * 58.6 \text{ MPa} = 1,119 \text{ kN/m}$$

Base shoe anchorage:

Typical rail section: 1,100mm high

$$M_p = 1.1 \text{ m} * P \text{ kN}; \quad P = \text{concentrated load on top rail}$$

$$M_t = 1.1 \text{ m} * U \text{ kN/m}; \quad U = \text{uniform load along top rail}$$

$$M_w = 1.1^2 \text{ m} / 2 * W \text{ kN/m}^2; \quad W = \text{Wind load on uniform load on glass}$$

Typical Anchor load – 300mm o.c. –

$$T_a = (M \text{ Nm/m}) * .30 \text{ m} / 0.03175 \text{ m} = 9.45 * M \text{ (N)}$$

For 12mm 304 SS ASTM E593 Condition AF cap screw to tapped steel

$$T_n = A_{\text{sn}} * t_c * 0.6 * F_{\text{tu}}$$

where  $t_c = 6.7 \text{ mm}$ ;  $A_{\text{sn}} = 26.117 \text{ mm}^2$  and  $F_{\text{tu}} = 400 \text{ MPa (58 ksi) (A36 steel plate)}$

$$\text{Plate thread stripping: } T_n = 26.117 \text{ mm}^2 * 6.7 \text{ mm} * 0.6 * 400 \text{ MPa} = 42.0 \text{ kN}$$

for screw thread stripping:  $A_{\text{ss}} = 17.064 \text{ mm}^2$  and  $F_{\text{tu}} = 571 \text{ MPa}$

$$T_n = 17.064 \text{ mm}^2 * 6.7 \text{ mm} * 0.6 * 571 \text{ MPa} = 39.17 \text{ kN}$$

Bolt tension strength =  $465.4 \text{ MPa} * 84.267 \text{ mm}^2 = 39.2 \text{ kN}$  bolt strength controls

Maximum service load:  $0.75 * 39.2 / 1.6 = 18.375 \text{ kN}$

Maximum allowable moment for 300mm on center spacing and direct bearing of base shoe on steel:

$$M_a = 18.375 \text{ kN} * [31.75 \text{ mm} - 0.5 * 18.375 \text{ kN} / (206.8 \text{ MPa} * 300 \text{ mm})] = 580.6 \text{ Nm per anchor}$$

$$M/m = 580.6 \text{ Nm} * 1/.3 = 1,935.6 \text{ Nm/m}$$

**ALLOWABLE LOADS FOR 1,100 mm RAIL HEIGHT**

$$P_a = 1,935.6 \text{ Nm} / 1.1 \text{ m} = 1.76 \text{ kN/m of glass}$$

$$U_a = 1,935.6 \text{ Nm/m} / 1.1 \text{ m} = 1.76 \text{ kN/m}$$

$$W_a = 2 * 1,935.6 \text{ Nm/m} / (1.1 \text{ m})^2 = 3.2 \text{ kN/m}^2$$

Side mounted base shoe:

Verify Anchor Pull through

For counter sunk screw

M12, 12mm dia

from ADM 5.4.2.2

$t_1 > 6\text{mm}$  so check with Eq 5.4.2.2-1 only

$$P_{nov} = Ct_1 F_{tu1} (D_{ws} - D_h)$$

$$P_{nov} = 1.0 * 15.1\text{mm} * 205\text{Mpa} (22\text{mm} - 12\text{mm}) = 30.955\text{kN}$$

$$T_a = 30,955/3 = 10,318\text{N}$$

For inset bolt

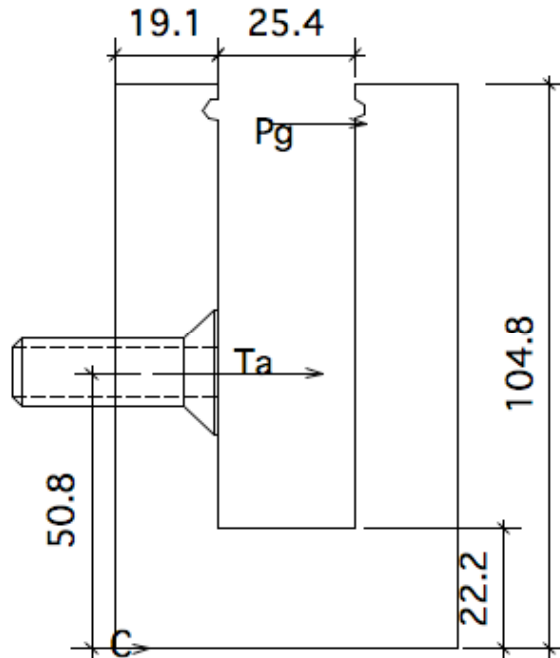
$$t_{min} = 6.35\text{mm}$$

$$P_{nov} = F_{tu}/\sqrt{3} * (A_v)$$

$$A_v = 6.35\text{mm} * \pi * 19.05\text{mm} = 380\text{mm}^2$$

$$P_{nov} = 205\text{MPa}/\sqrt{3} * (380\text{mm}^2) = 45\text{kN}$$

Screw or bolt strength = 39.2kN



For standard installation, 1,100mm guard height measured from top of base shoe:

Moment calculated about bottom of base shoe,  $H = 1,100\text{mm} + 105\text{mm} = 1,205\text{mm}$

$$M_p = 1.205\text{m} * P \text{ kN}; \quad P = \text{concentrated load on top rail}$$

$$M_t = 1.205\text{m} * U \text{ kN/m}; \quad U = \text{uniform load along top rail}$$

$$M_w = 1.205^2\text{m}^2/2 * W = 0.726 * W \text{ kN/m}^2; \quad W = \text{Wind load on uniform load on glass}$$

Typical Anchor load – 300mm o.c. –

$$T_a = (M_{Nm/m}) * .30\text{m}/0.050\text{m} = 6 * M \text{ (N)}$$

$$M_a = 10,318\text{N} * [50.8\text{mm} - 0.5 * 10,318\text{N}/(205\text{MPa} * 300\text{mm})] = 588\text{Nm per anchor}$$

$$M/m = 588\text{Nm} * 1/.3 = 1,960\text{Nm/m}$$

ALLOWABLE LOADS FOR 1,100 mm RAIL HEIGHT

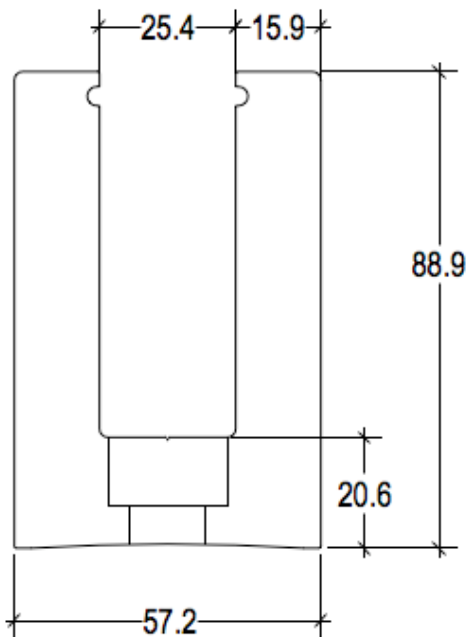
$$P_a = 1,960\text{Nm}/1.205\text{m} = 1.627\text{kN/m of glass}$$

$$U_a = 1,960\text{Nm/m}/1.205\text{m} = 1.627\text{kN/m}$$

$$W_a = 2 * 1,960\text{Nm/m}/(1.205\text{m})^2 = 2.7\text{kN/m}^2$$

**B5L Low Profile Base Shoe 57mm x 89mm**

6063-T52 Aluminum extrusion



Fully tempered glass glazed in place with Taper-Loc® dry-glaze.

Shoe strength – Vertical legs:

Glass reaction by bearing on legs to form couple.

Allowable moment on legs:

$$M_a = S_1 F_y$$

$$F_t = F_c = 86.18 \text{ MPa (ADM Table 2-23, Sec 3.4.4)}$$

$$S_1 = 1000 \text{ mm} * (15.9 \text{ mm})^2 / 6 = 42,135 \text{ mm}^3$$

$$M_a = 42,135 \text{ mm}^3 * 86.18 \text{ MPa} = 3.63 \text{ kNm/m}$$

Leg shear strength @ bottom

$$t_{\min} = 15.9 \text{ mm}$$

$$F_v = 58.6 \text{ MPa (ADM Table 2-24, Sec 3.4.20)}$$

$$V_{\text{all}} = 15.9 \text{ mm} * 1,000 * 58.6 \text{ MPa} = 931.7 \text{ kN/m}$$

Base shoe anchorage:

Typical rail section: 1,100mm high

$$M_p = 1.1 \text{ m} * P \text{ kN};$$

P = concentrated load on top rail

$$M_t = 1.1 \text{ m} * U \text{ kN/m};$$

U = uniform load along top rail

$$M_w = 1.1^2 \text{ m} / 2 * W \text{ kN/m}^2;$$

W = Wind load on uniform load on glass

Anchor 12mm Cap screws – 304 SS

Maximum allowable moment for 300mm on center spacing and direct bearing of base shoe on steel:

$$M = 18.375 \text{ kN} * [28.6 \text{ mm} - 0.5 * 18.375 \text{ kN} / (206.8 \text{ MPa} * 300 \text{ mm})] = 525.5 \text{ Nm per anchor}$$

$$M/m = 525.5 \text{ Nm} * 1/.3 = 1,751.7 \text{ Nm/m}$$

ALLOWABLE LOADS FOR 1,100 mm RAIL HEIGHT

$$P_a = 1,751.7 \text{ Nm} / 1.1 \text{ m} = 1.59 \text{ N/m of glass}$$

$$U_a = 1,751.7 \text{ Nm/m} / 1.1 \text{ m} = 1.59 \text{ kN/m}$$

$$W_a = 2 * 1,751.7 \text{ Nm/m} / (1.1 \text{ m})^2 = 2.9 \text{ kN/m}^2$$

Side mounted base shoe:

Verify Anchor Pull through

For counter sunk screw

M12, 12mm dia

from ADM 5.4.2.2

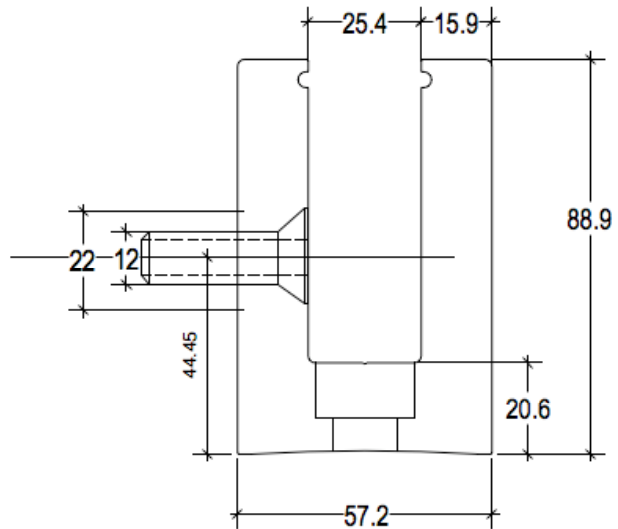
$t_1 > 6\text{mm}$  so check with Eq 5.4.2.2-1 only

$$P_{nov} = Ct_1 F_{tu1} (D_{ws} - D_h)$$

$$P_{nov} = 1.0 * 15.1\text{mm} * 205\text{Mpa} (22\text{mm} - 12\text{mm}) = 30.955\text{kN}$$

$$T_a = 30,955/3 = 10,318\text{N}$$

Screw or bolt strength = 39.2kN



For standard installation, 1,100mm guard height measured from top of base shoe:

Moment calculated about bottom of base shoe,  $H = 1,100\text{mm} + 89\text{mm} = 1,189\text{mm}$

$$M_p = 1.189\text{m} * P \text{ kN}; \quad P = \text{concentrated load on top rail}$$

$$M_l = 1.189\text{m} * U \text{ kN/m}; \quad U = \text{uniform load along top rail}$$

$$M_w = 1.189^2\text{m}^2/2 * W = 0.707 * W \text{ kN/m}^2; \quad W = \text{Wind load on uniform load on glass}$$

Typical Anchor load – 300mm o.c. –

$$T_a = (M\text{Nm/m}) * .30\text{m}/0.044\text{m} = 6.82 * M \text{ (N)}$$

$$M_a = 10,318\text{N} * [44.45\text{mm} - 0.5 * 10,318\text{N}/(205\text{MPa} * 300\text{mm})] = 458.6\text{Nm per anchor}$$

$$M/\text{m} = 458.6\text{Nm} * 1/.3 = 1,529\text{Nm/m}$$

ALLOWABLE LOADS FOR 1,100 mm RAIL HEIGHT

$$P_a = 1,529\text{Nm}/1.189\text{m} = 1.286\text{kN/m of glass}$$

$$U_a = 1,529\text{Nm/m}/1.189\text{m} = 1.286\text{kN/m}$$

$$W_a = 2 * 1,529\text{Nm/m}/(1.189\text{m})^2 = 2.16\text{kN/m}^2$$

**B5T Tapered Base Shoe**

6063-T52 Aluminum

Shoe strength – Vertical legs:

Glass reaction by bearing on legs to form couple.

Allowable moment on legs:

$$M_a = S_1 F_y$$

$$F_y = 86.18 \text{ MPa (ADM Table 2-23, Sec 3.4.4)}$$

$$S_1 = 1,000\text{mm} \times 12.66\text{mm}^2 / 6 = 26,712.6 \text{ mm}^3/\text{m}$$

$$M_a = 86.18 \text{ MPa} \times 26,712.6 \text{ mm}^3/\text{m} = 2.3\text{kNm}/\text{m}$$

Leg shear strength @ base

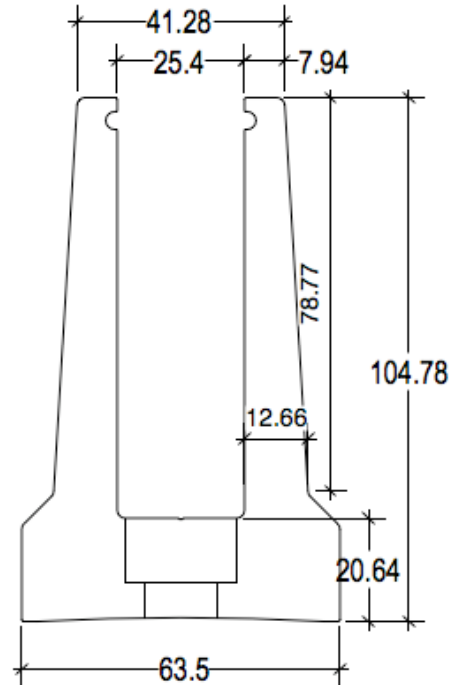
$$t_{\min} = 12.66\text{mm}$$

$$F_v = 37.92 \text{ MPa (ADM Table 2-24, Sec 3.4.20)}$$

$$V_{\text{all}} = 37.92\text{MPa} \times 12.66\text{mm} \times 1,000\text{mm}/\text{m} = 33 \text{ k/ft}$$

$$480\text{kN}/\text{m}$$

Can be anchored down same as the standard base shoe. The anchorage will have the same strength and loading characteristics. Determine allowable moment on base shoe based on anchorage strength:



Base shoe anchorage:

Typical rail section: 1,100mm high

$$M_p = 1.1\text{m} \times P \text{ kN};$$

P = concentrated load on top rail

$$M_t = 1.1\text{m} \times U \text{ kN}/\text{m};$$

U = uniform load along top rail

$$M_w = 1.1^2\text{m}^2/2 \times W \text{ kN}/\text{m}^2;$$

W = Wind load on uniform load on glass

Anchor 12mm Cap screws – 304 SS

Maximum allowable moment for 300mm on center spacing and direct bearing of base shoe on steel:

$$M = 18.375\text{kN} \times [31.75\text{mm} - 0.5 \times 18.375\text{kN} / (206.8\text{MPa} \times 300\text{mm})] = 580.7\text{Nm per anchor}$$

$$M/\text{m} = 580.7\text{Nm} \times 1/.3 = 1,935.6\text{Nm}/\text{m}$$

**ALLOWABLE LOADS FOR 1,100 mm RAIL HEIGHT**

$$P_a = 1,935.6\text{Nm}/1.1\text{m} = 1.76\text{N}/\text{m of glass}$$

$$U_a = 1,935.6\text{Nm}/\text{m}/1.1\text{m} = 1.76\text{kN}/\text{m}$$

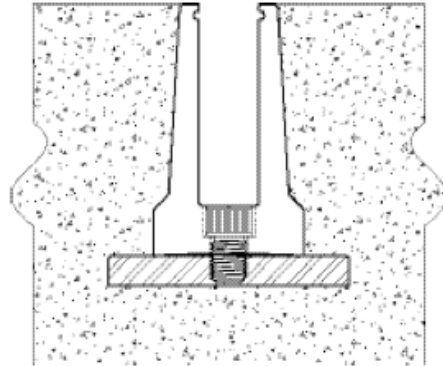
$$W_a = 2 \times 1,935.6\text{Nm}/\text{m} / (1.1\text{m})^2 = 3.2\text{kN}/\text{m}^2$$

Tapered base shoe cannot be side mounted.



Embedded Base Shoe Option (All base shoe types can be used). Check based on tapered base:

Strength of embed in concrete will be dependent on side cover, embed depth and concrete strength. Attachment to substrate is only required to secure the base shoe during concrete placement. When concrete strength is inadequate or side cover is too thin the base shoe anchorage shall be based on attachment to the substrate.



Example of checking base shoe embedment:

1.) Calculate reaction on concrete:

Compression on top edge:

$$0.85 * f'_c * a = M / (h - a/2)$$

Solve for a

$$1/2 a^2 - 0.85 f'_c h_a - M = 0$$

Where:

M = guard moment at the top of the base shoe

$h_a$  = lesser of base shoe height or embedment depth

$f'_c$  = concrete compressive strength

example:

M = 10,000 #''/ft, h = 4.125'',  $f'_c$  = 2,500 psi

$$1/2 a^2 - 0.85 * f'_c * h_a - M = 0$$

using the quadratic equation to solve for a:

$$[f'_c * h_a \pm \sqrt{(f'_c * h_a)^2 + 4 * 0.5 * M}] / (2 * 0.5) = a$$

if  $a < 1/3 * h_a$  the concrete strength and embed depth is adequate.

if  $a > 1/3 * h_a$  the concrete strength or (and) embed depth strength must be increased.

Check side cover for shear

$$1.6M \leq 0.85 * 0.1 f'_c * 1.0 * d^2 / 6$$

$$M \leq 0.00885 * f'_c * d^2$$

where d = side cover (meters)

When side cover is greater than  $2 * h_a$  it will not control allowable moment.

Side cover strength can be increased by using steel reinforcement bars.

**B5A SurfaceMate Square Base Shoe 63.5mm X 106mm**

B5A Shoe is designed to be interchangeable with the B5S shoe. The B5A base shoe allowable loads are the same as for the B5S shoes for all anchor types and configurations.

Refer to the B5S base shoe calculations for allowable loads for the anchor time.

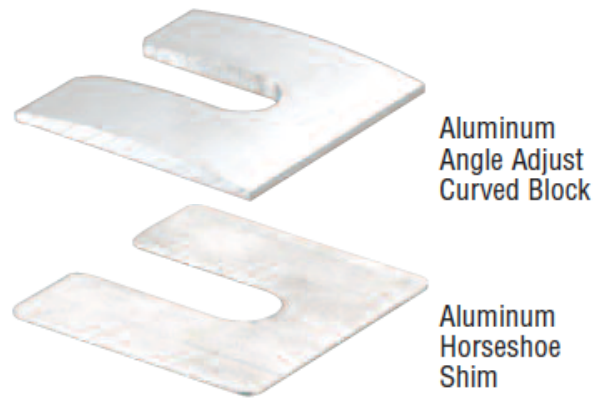
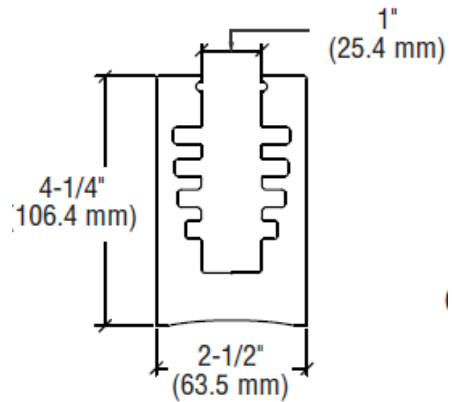
**Not to be used on wood substrates.**

**SurfaceMate Angle Adjust Curved Blocks**

Used at each anchor bolt to allow adjustment of the B5A base shoe to plumb on an out of level or uneven substrate.

When used on a steel substrate there is no reduction in the allowable loads.

When installed on a concrete substrate grout shall be packed solid under the base shoe or a continuous shim strip used in order to develop the full allowable loads.



When installed on concrete substrate without grouting or continuous shim the allowable loads are adjusted to:

For 86mm anchor edge distance

$$M_a = 9.39\text{kN} \cdot [31.9\text{mm} - 0.5 \cdot 9.39\text{kN} / (2 \cdot 0.85 \cdot 20,684.3\text{kN/m}^2 \cdot 57.5\text{mm})] = 277.7 \text{ kN}\cdot\text{mm}$$

Maximum allowable wind loads for 300mm spacing:

$$920\text{mm height: } w = (277.7 \cdot 2 / 920^2) / 0.3 = 2.188\text{kN/m}^2$$

$$1,100\text{mm height: } w = (277.7 \cdot 2 / 1100^2) / 0.3 = 1.53\text{kN/m}^2$$

For minimum edge distance = 60mm

$$\phi M_n = 7.64\text{kN} \cdot [31.9\text{mm} - 0.5 \cdot 7.64\text{kN} / (2 \cdot 0.85 \cdot 20,684.3\text{kN/m}^2 \cdot 57.5\text{mm})] = 229.3\text{kN}\cdot\text{mm}$$

Anchor spacing must be decreased for 1100mm guard height when 1.5kN-m live load applies.

$$S_{50-42} = 229.3\text{kN}\cdot\text{mm} / (1100 \cdot 1.5\text{kN}\cdot\text{m}) \cdot 1000 = 139\text{mm o.c}$$

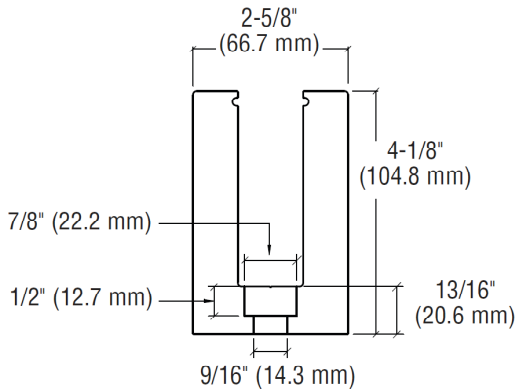
Maximum allowable wind loads (300mm o.c. spacing):

$$920\text{mm height: } w = (229.3 \cdot 2 / 920^2) / 0.3 = 1.8\text{kN/m}^2$$

$$1,100\text{mm height: } w = (229.3 \cdot 2 / 1100^2) / 0.3 = 1.26\text{kN/m}^2$$

**Not to be used on wood substrates.**

**B6S 2 5/8" X 4 1/8" GLASS BALUSTRADE BASE SHOE**



Heavy Duty Square Base Shoe

6063-T52 Aluminum extrusion

Fully tempered glass glazed in place, either wet glazing cement or GlassWedges®.

Shoe strength – Vertical legs:

Glass reaction by bearing on legs to form couple. Allowable moment on legs:

$$M_a = S_l F_y$$

$$F_t = F_c = 86.18 \text{ MPa (ADM Table 2-23, Sec 3.4.4)}$$

$$S_l = 1,000\text{mm} \times 19.16\text{mm}^2 / 6 = 61152\text{mm}^3/\text{m}$$

$$M_a = 86.18\text{MPa} \times 61152\text{mm}^3/\text{m} = 5.27 \text{ kN-m/m}$$

Leg shear strength @ groove

$$t_{\min} = 8.76\text{mm}$$

$$F_v = 37.92\text{MPa (ADM Table 2-24, Sec 3.4.20)}$$

$$V_{\text{all}} = 8.76\text{mm} \times 1000 \times 37.92\text{MPa} = 332.2 \text{ kN/m}$$

Base shoe anchorage:

Typical rail section: 1100mm high 0.75kN/m top rail load or 1.0kN/m<sup>2</sup> panel load

$$M_t = 1100 \times 0.75 = 825\text{kN-mm/m}$$

$$M_w = 1.0\text{kN/m}^2 \times 1.1\text{m} \times 1100\text{mm} / 2 = 605\text{kN-mm/m}$$

Typical Anchor load – 300mm o.c. –  $T_a = 825\text{kN-mm/m} / 33.46\text{mm} \times 0.3\text{m} = 7.40$

Maximum allowable moment for 12mm cap screws ( $T_a = 18.375\text{kN}$ ) 300mm on center spacing and direct bearing of base shoe on steel:

$$M_a = 18.375\text{kN} \times [33.46\text{mm} - 0.5 \times 18.375\text{kN} / (206.8\text{MP} \times 300\text{mm})] / .3\text{m} = 2.04\text{kN-mm/m}$$

**ALLOWABLE LOADS FOR 1,100 mm RAIL HEIGHT**

$$P_a = 2,040\text{Nm} / 1.1\text{m} = 1.85\text{kN/m of glass}$$

$$U_a = 2,040\text{Nm/m} / 1.1\text{m} = 1.85\text{kN/m}$$

$$W_a = 2 \times 2,040\text{Nm/m} / (1.1\text{m})^2 = 3.37\text{kN/m}^2$$

For anchor into concrete:

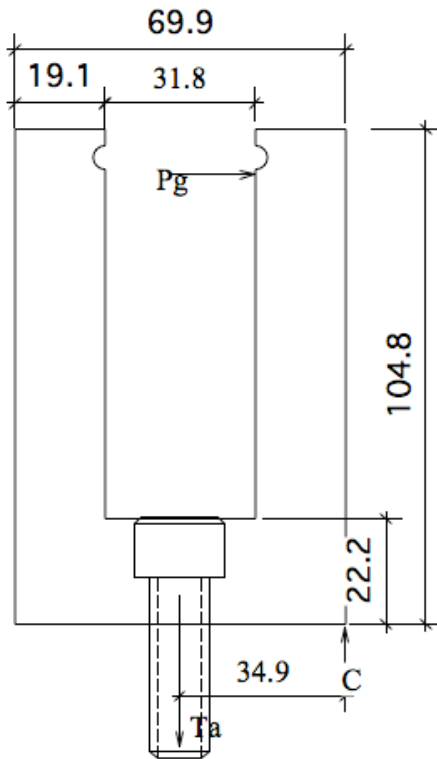
Hilti HSL-3 M8 x 3-3/4" anchor in accordance with ESR-1545.

**FASCIA (SIDE) MOUNTED BASE SHOE**

For side mounted base the allowable loads are the same as for the B5S base shoe.

Alternative anchors will provide the same allowable loads as for the B5S base shoe to steel, concrete or wood.

**B7S 70mm X 105 GLASS BALUSTRADE BASE SHOE**



Heavy Duty Square Base Shoe

6063-T52 Aluminum extrusion

Fully tempered glass glazed in place with Taper-Loc® dry-glaze.

Shoe strength – Vertical legs:

Glass reaction by bearing on legs to form couple.

Allowable moment on legs:

$$M_a = S_1 F_y$$

$$F_t = F_c = 86.18 \text{ MPa (ADM Table 2-23, Sec 3.4.4)}$$

$$S_1 = 1000 \text{ mm} * (19.1 \text{ mm})^2 / 6 = 60,802 \text{ mm}^3$$

$$M_a = 60,802 \text{ mm}^3 * 86.18 \text{ MPa} = 5.24 \text{ kNm/m}$$

Leg shear strength @ bottom

$$t_{\min} = 19.1 \text{ mm}$$

$$F_v = 58.6 \text{ MPa (ADM Table 2-24, Sec 3.4.20)}$$

$$V_{\text{all}} = 19.1 \text{ mm} * 1,000 * 58.6 \text{ MPa} = 1,119 \text{ kN/m}$$

Base shoe anchorage:

Typical rail section: 1,100mm high

$$M_p = 1.1 \text{ m} * P \text{ kN};$$

P = concentrated load on top rail

$$M_t = 1.1 \text{ m} * U \text{ kN/m};$$

U = uniform load along top rail

$$M_w = 1.1^2 \text{ m}^2 / 2 * W \text{ kN/m}^2;$$

W = Wind load on uniform load on glass

Anchor 12mm Cap screws – 304 SS

Maximum allowable moment for 300mm on center spacing and direct bearing of base shoe on steel:

$$M = 18.375 \text{ kN} * [34.9 \text{ mm} - 0.5 * 18.375 \text{ kN} / (206.8 \text{ MPa} * 300 \text{ mm})] = 638.6 \text{ Nm per anchor}$$

$$M/m = 638.6 \text{ Nm} * 1/.3 = 2,128.7 \text{ Nm/m}$$

ALLOWABLE LOADS FOR 1,100 mm RAIL HEIGHT

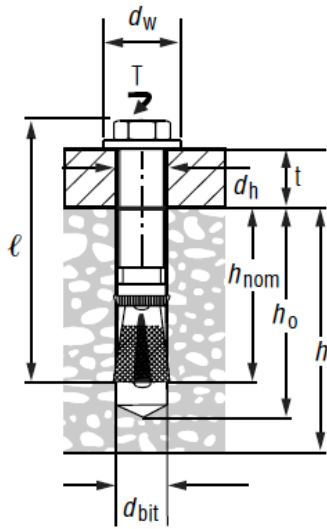
$$P_a = 2,128.7 \text{ Nm} / 1.1 \text{ m} = 1.94 \text{ kN/m of glass}$$

$$U_a = 2,128.7 \text{ Nm/m} / 1.1 \text{ m} = 1.94 \text{ kN/m}$$

$$W_a = 2 * 2,128.7 \text{ Nm/m} / (1.1 \text{ m})^2 = 3.52 \text{ kN/m}^2$$

For side mounted base the allowable loads are the same as for the B5S base shoe.

Standard Concrete Anchor:  
 Hilti HSL-3 Heavy Duty Sleeve  
 Anchor  
 Zinc -plated carbon steel.  
 Try M8 size for clearance  
 requirements in base shoe.  
 Design values from ESR-1545  
 Complies with ICC-AC193



**Table 1 — HSL-3 Specifications**

Details			M8	
nominal drill bit diameter <sup>1</sup>	$d_{bit}$	mm	12	
Hilti matched-tolerance carbide-tipped drill bit	-	-	TE-CX 12/22 TE-YX 12/35	
minimum base material thickness (to obtain smallest critical edge distance)	$h$	mm (in.)	110 (120) 4 3/8 (4-3/4)	
minimum hole depth	$h_o$	mm (in.)	80 (3-1/8)	
effective embedment depth	$h_{ef,min}$	mm (in.)	60 (2-3/8)	
minimum clearance hole diameter in part being fastened	$d_h$	mm (in.)	14 (1/2)	
max. cumulative gap between part(s) being fastened and concrete surface	-	mm (in.)	4 (1/8)	
maximum thickness of part fastened HSL-3, HSL-3-B	$t$	mm (in.)	20 (3/4)	40 (1-1/2)
overall length of anchor HSL-3, HSL-3-B	-	mm (in.)	98 (3-7/8)	118 (4-5/8)
maximum thickness of part fastened HSL-3-G	$t$	mm (in.)	20 (3/4)	
overall length of anchor HSL-3-G	-	mm (in.)	102 (4)	
washer diameter	$d_w$	mm (in.)	20 (3/4)	
installation torque HSL-3	$T_{inst}$	Nm (ft-lb)	25 (18)	
installation torque HSL-3-G	$T_{inst}$	Nm (ft-lb)	20 (15)	
wrench size HSL-3, HSL-3-G	-	mm	13	
wrench size HSL-3-B	-	mm		

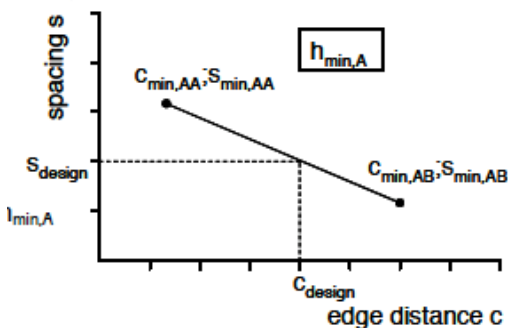
<sup>1</sup> Use metric bits only.

Allowable tension load for  $f'_c = 13.8$  MPa (2,000 psi) concrete, uncracked, case B:

$T = 7.767$  kN (1,746#)

Anchor spacing = 300mm

Edge distance > 60mm (2-3/8")



B	Minimum concrete thickness	$h_{min,B}$ <sup>4</sup>	in. (mm)	4-3/8 (110)
B	Critical edge distance <sup>2</sup>	$C_{cr,B}$	in. (mm)	5-7/8 (150)
B	Minimum edge distance <sup>3</sup>	$C_{min,BA}$	in. (mm)	2-3/8 (60)
B	Minimum anchor spacing <sup>3</sup>	$S_{min,BA}$	in. (mm)	7 (180)
B	Minimum edge distance <sup>3</sup>	$C_{min,BB}$	in. (mm)	4 (100)
B	Minimum anchor spacing <sup>3</sup>	$S_{min,BB}$	in. (mm)	2-3/8 (60)

Will plot above and to right of line therefore can use full allowable tension load.

**Table 5—HSL-3 Allowable Static Tension (ASD), Normal Weight Uncracked Concrete (pounds)<sup>1,3</sup>**

Nominal Anchor Diameter	Embedment Depth $h_{ef}$		Concrete Compressive Strength <sup>2</sup>							
			f'c = 2,000 psi		f'c = 3,000 psi		f'c = 4,000 psi		f'c = 6,000 psi	
	mm	in.	Condition A	Condition B	Condition A	Condition B	Condition A	Condition B	Condition A	Condition B
M8	60	2.36	1,746	1,746	2,139	2,139	2,470	2,470	3,025	3,025
M10	70	2.76	2,631	2,280	3,222	2,792	3,720	3,224	4,556	3,949
M12	80	3.15	3,214	2,785	3,936	3,411	4,545	3,939	5,567	4,825
M16	100	3.94	4,492	3,893	5,501	4,768	6,352	5,505	7,780	6,743
M20	125	4.92	6,277	5,440	7,688	6,663	8,877	7,694	10,873	9,423
M24	150	5.91	8,252	7,152	10,106	8,759	11,670	10,114	14,292	12,387

For SI: 1 lbf = 4.45 N, 1 psi = 0.006895 MPa For pound-inch units: 1 mm = 0.03937 inches

<sup>1</sup>Values are for single anchors with no edge distance or spacing reduction. For other cases, see Section 4.2 Eq. (5).

<sup>2</sup>Values are for normal weight concrete. For sand-lightweight concrete, multiply values by 0.85. For all-lightweight concrete, multiply values by 0.75. See ACI 318-02 D.3.4.

<sup>3</sup>Condition A applies where the potential concrete failure surfaces are crossed by supplementary reinforcement proportioned to tie the potential concrete failure prism into the structural member. Condition B applies where such supplementary reinforcement is not provided, or where pullout or pryout strength governs.

**Table 6—HSL-3 Allowable Static Tension (ASD,) Normal Weight Cracked Concrete (pounds)<sup>1,3</sup>**

Nominal Anchor Diameter	Embedment Depth $h_{ef}$		Concrete Compressive Strength <sup>2</sup>							
			f'c = 2,000 psi		f'c = 3,000 psi		f'c = 4,000 psi		f'c = 6,000 psi	
	mm	in.	Condition A	Condition B	Condition A	Condition B	Condition A	Condition B	Condition A	Condition B
M8	60	2.36	1,167	1,167	1,429	1,429	1,650	1,650	2,021	2,021
M10	70	2.76	1,867	1,867	2,286	2,286	2,640	2,640	3,233	3,233
M12	80	3.15	3,214	2,785	3,936	3,411	4,545	3,939	5,567	4,825
M16	100	3.94	4,492	3,893	5,501	4,768	6,352	5,505	7,780	6,743
M20	125	4.92	6,277	5,440	7,688	6,663	8,877	7,694	10,873	9,423
M24	150	5.91	8,252	7,152	10,106	8,759	11,670	10,114	14,292	12,387

For SI: 1 lbf = 4.45 N, 1 psi = 0.006895 MPa For pound-inch units: 1 mm = 0.03937 inches

<sup>1</sup>Values are for single anchors with no edge distance or spacing reduction. For other cases, see Section 4.2 Eq. (5).

<sup>2</sup>Values are for normal weight concrete. For sand-lightweight concrete, multiply values by 0.85. For all-lightweight concrete, multiply values by 0.75. See ACI 318-02 D.3.4.

<sup>3</sup>Condition A applies where the potential concrete failure surfaces are crossed by supplementary reinforcement proportioned to tie the potential concrete failure prism into the structural member. Condition B applies where such supplementary reinforcement is not provided, or where pullout or pryout strength governs.

### Conclusion:

Hilti HSL-3 Heavy Duty Sleeve Anchor will work to anchor the base shoe to concrete in uncracked concrete with a minimum strength of 13.8 MPa (2,000 psi) and cracked concrete with a minimum strength of 27.6 MPa (4,000 psi).

**SPECIAL INSPECTION IS REQUIRED.**

Allowable base shoe moment:

Standard and Tapered shoes:

$$a = 7,767\text{N} / (.3 * 0.85 * 2 * 13,800,000\text{Pa}) = 0.0011\text{m}$$

$$M_a = [7,767\text{N} * (d-a) / 2] * (1/0.3) = 12,945\text{N}(d-a)$$

For standard base shoe  $d = 63.5\text{mm}$

$$M_a = 12,945\text{N}(0.0635 - 0.0011) = 807.8\text{Nm/m}$$

**ALLOWABLE LOADS USING HILTI HSL-3 M8 ANCHORS AT 300MM ON CENTER**

**Standard base shoe 64mm x105mm continued**

ALLOWABLE LOADS FOR 1,100 mm RAIL HEIGHT

$$P_a = 807.8\text{Nm}/1.1\text{m} = 0.73\text{kN/m of glass}$$

$$U_a = 807.8\text{Nm}/1.1\text{m} = 0.73\text{kN/m}$$

$$W_a = 2*807.8\text{Nm}/(1.1\text{m})^2 = 1.335\text{kN/m}^2$$

Same allowable loads are applicable to the **tapered base shoe**

**Heavy base shoe 70mm x105mm**

$$d = 69.9\text{mm}$$

$$M_a = 12,945\text{N}(0.069-0.0011) = 879.0\text{Nm/m}$$

ALLOWABLE LOADS FOR 1,100 mm RAIL HEIGHT

$$P_a = 879.0\text{Nm}/1.1\text{m} = 0.799\text{kN/m of glass}$$

$$U_a = 879.0\text{Nm}/1.1\text{m} = 0.799\text{kN/m}$$

$$W_a = 2*879.0\text{Nm}/(1.1\text{m})^2 = 1.453\text{kN/m}^2$$

**Low Profile Base Shoe 57mm x 89mm**

For standard base shoe  $d = 57.2\text{mm}$

$$M_a = 12,945\text{N}(0.0572-0.0011) = 726.2\text{Nm/m}$$

ALLOWABLE LOADS FOR 1,100 mm RAIL HEIGHT

$$P_a = 726.2\text{Nm}/1.1\text{m} = 0.66\text{kN/m of glass}$$

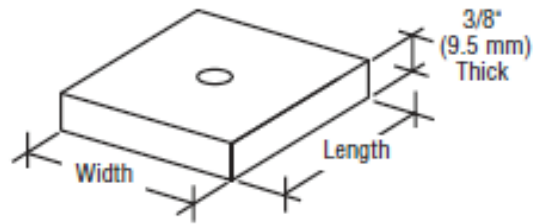
$$U_a = 726.2\text{Nm}/1.1\text{m} = 0.66\text{kN/m}$$

$$W_a = 2*726.2\text{Nm}/(1.1\text{m})^2 = 1.2\text{kN/m}^2$$

**Drain Blocks**

Drain blocks may be used under the base shoe to provide a water drainage path on exterior decks.

When used on steel substrate there is no reduction in the allowable loads.

**Not to be used on wood substrate.**

When used on concrete the allowable loads are reduced as follows:

**B5S, B5T and B5A base shoes: 64mm x 57mm**

For 96mm anchor edge distance

$$M_a = 9.39\text{kN} \cdot [31.92 - 0.5 \cdot 9.39\text{kN} / (2 \cdot 0.85 \cdot 20.68\text{MPa} \cdot 57.5)] / 0.3\text{m} = 926.4\text{N} \cdot \text{m} / \text{m}$$

Maximum allowable loads for 300mm spacing:

$$P_a = 926.4\text{Nm} / 1.1\text{m} = 0.84\text{kN} / \text{m} \text{ of glass}$$

$$U_a = 926.4\text{Nm} / \text{m} / 1.1\text{m} = 0.84\text{kN} / \text{m}$$

$$W_a = 2 \cdot 926.4\text{Nm} / \text{m} / (1.1\text{m})^2 = 1.53\text{kN} / \text{m}^2$$

For minimum edge distance is 60mm

$$\phi M_n = 7.638\text{kN} \cdot [31.92 - 0.5 \cdot 7.638\text{kN} / (2 \cdot 0.85 \cdot 20.68\text{MPa} \cdot 57.5)] / 0.3\text{m} = 764.6\text{N} \cdot \text{m} / \text{m}$$

Maximum allowable loads for 300mm spacing:

$$P_a = 764.6\text{Nm} / 1.1\text{m} = 0.695\text{kN} / \text{m} \text{ of glass}$$

$$U_a = 764.6\text{Nm} / \text{m} / 1.1\text{m} = 0.695\text{kN} / \text{m}$$

$$W_a = 2 \cdot 764.6\text{Nm} / \text{m} / (1.1\text{m})^2 = 1.264\text{kN} / \text{m}^2$$

Anchor spacing must be decreased for 1100mm guard height when 0.74kN/m live load applies.

$$S_{50-42} = 300\text{mm} \cdot 0.695 / 0.74 = 280\text{mm o.c. (use 11 anchors for 3m section)}$$

**B5L base shoe: 57mm x 64mm**

For 96mm anchor edge distance

$$M_a = 9.39\text{kN} \cdot [28.73 - 0.5 \cdot 9.39 / (2 \cdot 0.85 \cdot 20.68\text{MPa} \cdot 63.85)] / .3 = 833.8\text{N} \cdot \text{m} / \text{m}$$

Maximum allowable loads for 300mm spacing:

$$P_a = 833.8\text{Nm} / 1.1\text{m} = 0.758\text{kN} / \text{m} \text{ of glass}$$

$$U_a = 833.8\text{Nm} / \text{m} / 1.1\text{m} = 0.758\text{kN} / \text{m}$$

$$W_a = 2 \cdot 833.8\text{Nm} / \text{m} / (1.1\text{m})^2 = 1.378\text{kN} / \text{m}^2$$

For minimum edge distance is 60mm

$$\phi M_n = 7.638\text{kN} \cdot [28.73 - 0.5 \cdot 7.638\text{kN} / (2 \cdot 0.85 \cdot 20.68\text{MPa} \cdot 63.85)] / 0.3\text{m} = 688.1\text{N} \cdot \text{m} / \text{m}$$

Maximum allowable loads for 300mm spacing:

$$P_a = 688.1\text{Nm} / 1.1\text{m} = 0.626\text{kN} / \text{m} \text{ of glass}$$

$$U_a = 688.1\text{Nm} / \text{m} / 1.1\text{m} = 0.626\text{kN} / \text{m}$$

$$W_a = 2 \cdot 688.1\text{Nm} / \text{m} / (1.1\text{m})^2 = 1.137\text{kN} / \text{m}^2$$



**B5L base shoe: 57mm x 64mm (cont)**

Anchor spacing must be decreased for 1100mm guard height when 0.74kN/m live load applies.

$$S_{50-42} = 300\text{mm} * 0.626 / 0.74 = 253\text{mm o.c. (use 12 anchors for 3m section)}$$

**B6S base shoe: 67mm x 73.4mm**

For 96mm anchor edge distance

$$M_a = 9.39\text{kN} * [33.46 - 0.5 * 9.39 / (2 * 0.85 * 20.68\text{MPa} * 73.4)] / .3 = 990\text{N-m/m}$$

Maximum allowable loads for 300mm spacing:

$$P_a = 990\text{Nm} / 1.1\text{m} = 0.9\text{kN/m of glass}$$

$$U_a = 990\text{Nm/m} / 1.1\text{m} = 0.9\text{kN/m}$$

$$W_a = 2 * 990\text{Nm/m} / (1.1\text{m})^2 = 1.636\text{kN/m}^2$$

For minimum edge distance is 60mm

$$M_a = 7.638\text{kN} * [33.46 - 0.5 * 7.638 / (2 * 0.85 * 20.68\text{MPa} * 73.4)] / .3 = 814.2\text{N-m/m}$$

Maximum allowable loads for 300mm spacing:

$$P_a = 814.2\text{Nm} / 1.1\text{m} = 0.74\text{kN}$$

$$U_a = 814.2\text{Nm/m} / 1.1\text{m} = 0.74\text{kN/m}$$

$$W_a = 2 * 814.2\text{Nm/m} / (1.1\text{m})^2 = 1.346\text{kN/m}^2$$

**B7S base shoe: 73.4mm x 67mm**

For 96mm anchor edge distance

$$M_a = 9.39\text{kN} * [36.7 - 0.5 * 9.39 / (2 * 0.85 * 20.68\text{MPa} * 67)] / .3 = 1,086\text{N-m/m}$$

Maximum allowable loads for 300mm spacing:

$$P_a = 1,086\text{Nm} / 1.1\text{m} = 0.988\text{kN/m of glass}$$

$$U_a = 1,086\text{Nm/m} / 1.1\text{m} = 0.988\text{kN/m}$$

$$W_a = 2 * 1,086\text{Nm/m} / (1.1\text{m})^2 = 1.795\text{kN/m}^2$$

For minimum edge distance is 60mm

$$M_a = 7.638\text{kN} * [36.7 - 0.5 * 7.638 / (2 * 0.85 * 20.68\text{MPa} * 67)] / .3 = 893.1\text{N-m/m}$$

Maximum allowable loads for 300mm spacing:

$$P_a = 893.1\text{Nm} / 1.1\text{m} = 0.812\text{kN}$$

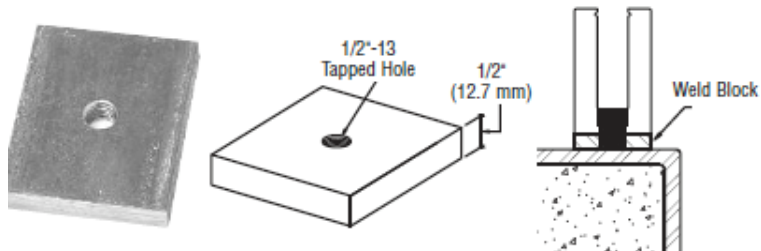
$$U_a = 893.1\text{Nm/m} / 1.1\text{m} = 0.812\text{kN/m}$$

$$W_a = 2 * 893.1\text{Nm/m} / (1.1\text{m})^2 = 1.476\text{kN/m}^2$$

**WELD BLOCKS:**

When attaching the base shoe to the appropriate steel weld blocks the strength shall be the same as for the base shoe attachment to steel substrate.

Weld block size shall be matched to the base shoe width.

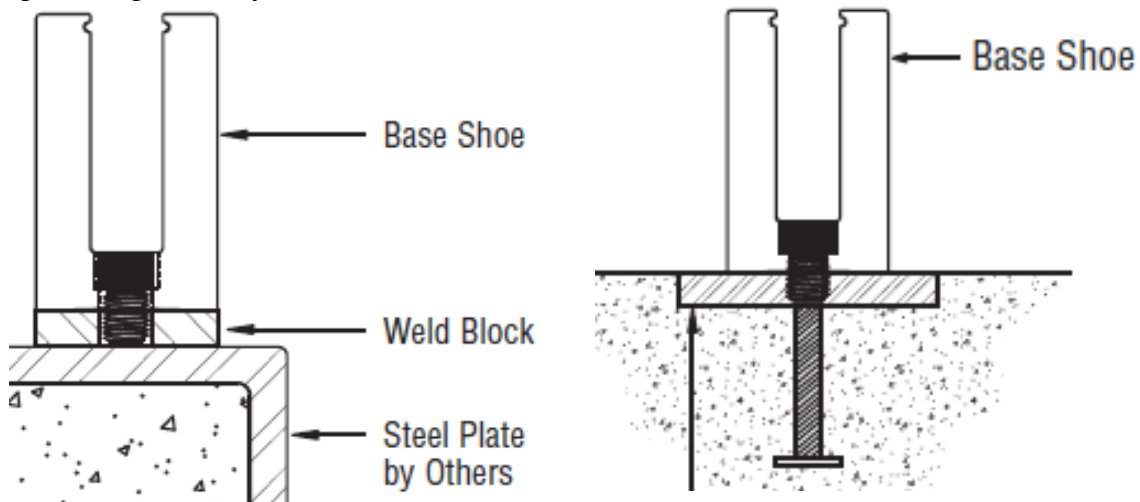


**Surface Mounting Base Shoes to Wood Decks:**

The base shoe develops a couple between the anchor tension and compression between the base shoe edge and the substrate to resist horizontal loads. Wood doesn't have adequate bearing compressive strength to reliably develop the requisite compressive strength when surface mounted. The shoe may be initially installed tight and appear to perform adequately; but cyclic loading will cause permanent deformation of the wood surface and loss of anchor pretension. This will result in rotation of the base shoe and increased couple forces resulting in excessive guard deflections and possible failure. For this reason the base shoes shall not be surface mounted directly to wood.

It is recommended that whenever possible the base shoe should use the fascia mount when attaching to wood.

When surface mounting to wood a steel or aluminum bar or angle shall be installed on the wood surface first. The bar or angle shall be designed to safely transfer the imposed loads from the base shoe to the wood deck. Attachment to the bar or angle shall be as specified previously.



Steel angle or plate bolted to wood deck with base shoe anchored to the plate or angle using 12mm cap screws into threaded weld blocks or tapped holes.

**CAP RAILS**

Guard applications require a top rail or handrail. The rail shall have adequate strength to support the live load of 200 lb concentrated or 50 plf distributed load assuming the failure of one glass lite at the location of the loading.

**GR 15 SERIES CAP RAIL**Area: 213.9 mm<sup>2</sup>I<sub>xx</sub>: 22,843.8 mm<sup>4</sup>I<sub>yy</sub>: 30,595.8 mm<sup>4</sup>r<sub>xx</sub>: 10.33mmr<sub>yy</sub>: 11.96mmC<sub>xx</sub>: 18.13mmC<sub>yy</sub>: 19.05mmS<sub>xx</sub>: 1,260 mm<sup>3</sup>S<sub>yy</sub>: 1,604 mm<sup>3</sup>

t = 1.27mm

Allowable stresses:

For stainless steel options: design using SEI/ASCE 8-02

From Table A1, F<sub>y</sub> = 344.7MPa for 1/4 hard 304 stainless steel cold formed sheet

$$F_{cr} = \frac{\pi^2 k \eta E_0}{12(1-\mu^2)(w/t)^2} \quad (\text{eq 3.3.1.1-9})$$

η = 0.49 (from table A8a)

k = 3(I<sub>s</sub>/I<sub>a</sub>)<sup>1/3</sup> + 1 < 4.0 = 4.0 for circular shape

μ = 0.3

E<sub>0</sub> = 186,200 MPa

$$F_{cr} = \frac{\pi^2 * 4.0 * 0.49 * 186,200 \text{MPa}}{12(1-0.3^2)(35.6/1.27)^2} = 419.8 \text{MPa but } \leq F_y$$

M<sub>n</sub> = S<sub>e</sub>F<sub>y</sub> = 1,260 mm<sup>3</sup>\*344.7MPa = 434.3Nm Vertical loading1,604 mm<sup>3</sup>\*344.7MPa = 552.9Nm Horizontal load

Use load factor of 1.6 and resistance factor of 1.0 (round shape using section modulus)

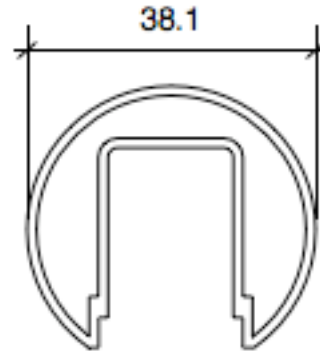
Determine allowable rail loads(ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights (M = w<sup>2</sup>/10)Vertical → uniform → w = (434.3Nm • 10 / (1.6\*(1.5m)<sup>2</sup>)) = 1,206 N/m

concentrated → P = 434.3Nm\*5 / (1.6\*1.5m) = 905N

Horizontal → uniform → w = (552.9Nm • 10 / (1.6\*(1.5m)<sup>2</sup>)) = 1,536 N/m

concentrated → P = 552.9Nm\*5 / (1.6\*1.5m) = 1,152N



GR 15 SERIES CAP RAIL For Brass:

Alloy C26000, Cartridge Brass, 70% Cu, 30% Zn

Cap rail fabricated from cold rolled sheet

$$F_{yu} \geq 296.5 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 k \eta E_0}{12(1-\mu^2)(w/t)^2}$$

$$\eta = 0.49$$

$$k = 3(I_s/I_a)^{1/3} + 1 < 4.0 = 4.0 \text{ for circular shape}$$

$$\mu = 0.34$$

$$E_0 = 116.5 \times 10^3 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 * 4.0 * 0.49 * 116.5 \times 10^3 \text{ MPa}}{12(1-0.34^2)(35.6/1.27)^2} = 280.6 \text{ MPa (40.7 ksi) but } \leq F_y$$

$$M_n = S_e F_y = 1,260 \text{ mm}^3 * 280.6 \text{ MPa} = 353.6 \text{ Nm} \quad \text{Vertical loading}$$

$$1,604 \text{ mm}^3 * 280.6 \text{ MPa} = 450.1 \text{ Nm} \quad \text{Horizontal load}$$

Use load factor of 1.6 and resistance factor of 1.0 (round shape using section modulus)

Determine allowable rail loads (ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights ( $M = wl^2/10$ )

$$\text{Vertical} \rightarrow \text{uniform} \rightarrow w = (353.6 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 982 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 353.6 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 736.7 \text{ N}$$

$$\text{Horizontal} \rightarrow \text{uniform} \rightarrow w = (450.1 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 1,250.3 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 450.1 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 937.7 \text{ N}$$

### Connector Sleeves

The sleeves fit tight (radial compression required) inside the rail and are secured with adhesive. The sleeve provides shear transfer between rail sections, vertically and horizontally. The sleeves can be used to connect straight or curved rail sections to corners and other rail sections.



Minimum shear strength of connectors:

For stainless steel:

$$F_{yv} = 289.6\text{MPa (42 ksi)}$$

$$t = 1.27\text{mm, } h = 74.9\text{mm (for 38mm rail)}$$

$$V_n = 4.84E_o t^3 (G_s/G_o)/h; \phi = 0.85$$

$$G_s/G_o = 0.90; E_o = 165.5\text{GPa (24,000 ksi)}$$

$$V_n = 4.84 * 165,500\text{MPa} * (1.27\text{mm})^3 (0.90) / 74.9\text{mm} = 19.7\text{kN controls}$$

$$\text{or } V_n = 0.95 * (289.6\text{MPa} * 1.27\text{mm} * 74.9\text{mm}) = 26.17\text{kN}$$

$$V_s = \phi V_n / 1.6 = 0.85 * 19.7\text{kN} / 1.6 = 10.47\text{kN}$$

For Brass:

$$F_{yv} = 172.4\text{MPa (25 ksi)}$$

$$t = 1.27\text{mm, } h = 74.9\text{mm (for 38mm rail)}$$

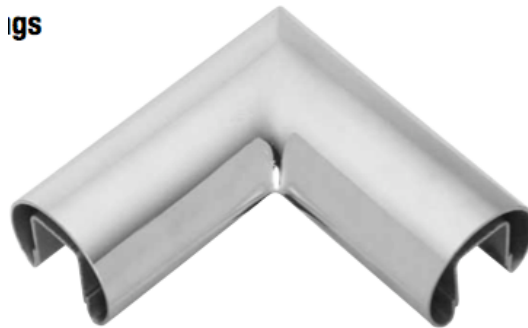
$$V_n = 0.95 * (172.4\text{MPa} * 1.27\text{mm} * 74.9\text{mm}) = 15.58\text{kN controls}$$

$$V_s = \phi V_n / 1.6 = 0.85 * 15.58\text{kN} / 1.6 = 8.28\text{kN}$$

### Welded Corners

Constructed from the standard rail sections. Corners are welded all around full thickness of metal.

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Load on corner is limited to shear and tension at corner.

Shear strength is same as the connector sleeve (weld length is same as connector perimeter)

$$\text{Tension: } = 1/0.6 * V = 1.667V$$

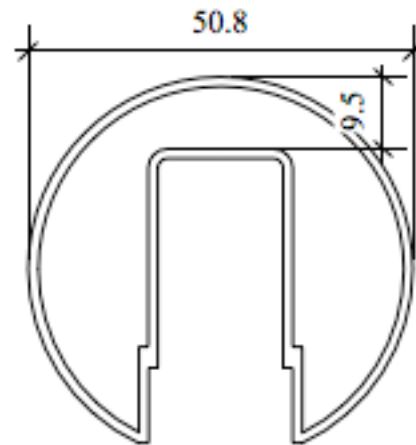
$$T_{ss} = 1.667 * 10.47\text{kN} = 17.45\text{kN}$$

$$T_{br} = 1.667 * 8.28\text{kN} = 13.80\text{kN}$$

**CRL GR 20 SERIES CAP RAIL**

Used as the top rail on glass balustrade panel guardrails

Area: 293.9 mm<sup>2</sup>  
 I<sub>xx</sub>: 62,393.4 mm<sup>4</sup>  
 I<sub>yy</sub>: 68,860.8 mm<sup>4</sup>  
 r<sub>xx</sub>: 14.57mm  
 r<sub>yy</sub>: 11.96mm  
 C<sub>xx</sub>: 24.90mm  
 C<sub>yy</sub>: 25.4mm  
 S<sub>xx</sub>: 2,505.9 mm<sup>3</sup>  
 S<sub>yy</sub>: 2,655.3 mm<sup>3</sup>  
 t = 1.27mm



Allowable stresses:

For stainless steel options: design using SEI/ASCE 8-02

From Table A1, 344.7MPa for 1/4 hard A304 stainless steel sheet used to form the rail.

$$F_{cr} = \frac{\pi^2 k \eta E_0}{12(1-\mu^2)(w/t)^2} \quad (\text{eq 3.3.1.1-9})$$

$$\eta = 0.49 \text{ (from table A8a)}$$

$$k = 3(I_s/I_a)^{1/3} + 1 < 4.0 = 4.0 \text{ for circular shape}$$

$$\mu = 0.3$$

$$E_0 = 186,200 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 * 4.0 * 0.49 * 186,200 \text{ MPa}}{12(1-0.3^2)(35.6/1.27)^2} = 419.8 \text{ MPa but } \leq F_y$$

$$M_n = S_e F_y = 2,505.9 \text{ mm}^3 * 344.7 \text{ MPa} = 863.8 \text{ Nm Vertical loading}$$

$$2,655.3 \text{ mm}^3 * 344.7 \text{ MPa} = 915.3 \text{ Nm Horizontal load}$$

Use load factor of 1.6 and resistance factor of 1.0 (round shape using section modulus)

Determine allowable rail loads (ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights ( $M = wl^2/10$ )

$$\text{Vertical} \rightarrow \text{uniform} \rightarrow w = (863.8 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 2,399 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 863.8 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 1,800 \text{ N}$$

$$\text{Horizontal} \rightarrow \text{uniform} \rightarrow w = (915.3 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 2,542 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 915.3 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 1,907 \text{ N}$$

GR 20 SERIES CAP RAIL For Brass:

Alloy C26000, Cartridge Brass, 70% Cu, 30% Zn

Cap rail fabricated from cold rolled sheet

$$F_{yu} \geq 296.5 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 k \eta E_0}{12(1-\mu^2)(w/t)^2}$$

$$\eta = 0.49$$

$$k = 3(I_s/I_a)^{1/3} + 1 < 4.0 = 4.0 \text{ for circular shape}$$

$$\mu = 0.34$$

$$E_0 = 116.5 \times 10^3 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 * 4.0 * 0.49 * 116.5 \times 10^3 \text{ MPa}}{12(1-0.34^2)(35.6/1.27)^2} = 280.6 \text{ MPa (40.7 ksi) but } \leq F_y$$

$$M_n = S_e F_y = 2,505.9 \text{ mm}^3 * 280.6 \text{ MPa} = 703.2 \text{ Nm} \quad \text{Vertical loading}$$

$$2,655.3 \text{ mm}^3 * 280.6 \text{ MPa} = 745.1 \text{ Nm} \quad \text{Horizontal load}$$

Use load factor of 1.6 and resistance factor of 1.0 (round shape using section modulus)

Determine allowable rail loads (ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights ( $M = wl^2/10$ )

$$\text{Vertical} \rightarrow \text{uniform} \rightarrow w = (703.2 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 1,953 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 703.2 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 1,465 \text{ N}$$

$$\text{Horizontal} \rightarrow \text{uniform} \rightarrow w = (745.1 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 2,069.7 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 745.1 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 1,552.3 \text{ N}$$

### Connector Sleeves

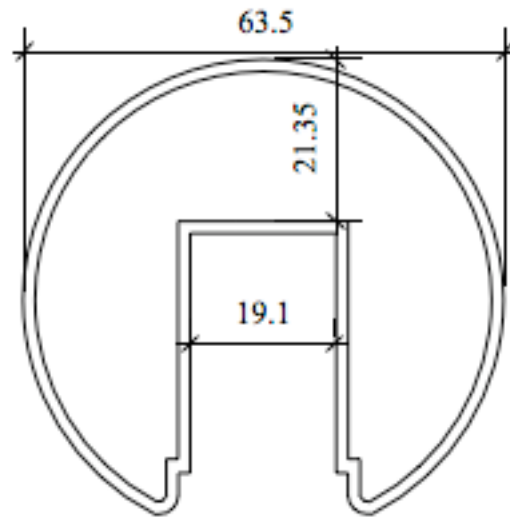
#### Corners

The connector sleeves and corners are demonstrated as adequate based on strength for the 38mm size.

**CRL GR 25 SERIES CAP RAIL**

Used as the top rail on glass balustrade panel guardrails

Area: 423.4 mm<sup>2</sup>  
 I<sub>xx</sub>: 138,449 mm<sup>4</sup>  
 I<sub>yy</sub>: 161,110.9 mm<sup>4</sup>  
 r<sub>xx</sub>: 18.08mm  
 r<sub>yy</sub>: 19.51mm  
 C<sub>xx</sub>: 30.81mm  
 C<sub>yy</sub>: 31.75mm  
 S<sub>xx</sub>: 4,494.0 mm<sup>3</sup>  
 S<sub>yy</sub>: 5,073.0 mm<sup>3</sup>  
 t = 1.27mm



Allowable stresses:

For stainless steel options: design  
 using SEI/ASCE 8-02

From Table A1, 344.7MPa for 1/4 hard A304 stainless steel sheet used to form the rail.

$$F_{cr} = \frac{\pi^2 k \eta E_0}{12(1-\mu^2)(w/t)^2} \quad (\text{eq 3.3.1.1-9})$$

$$\eta = 0.49 \quad (\text{from table A8a})$$

$$k = 3(I_s/I_a)^{1/3} + 1 < 4.0 = 4.0 \quad \text{for circular shape}$$

$$\mu = 0.3$$

$$E_0 = 186,200 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 * 4.0 * 0.49 * 186,200 \text{ MPa}}{12(1-0.3^2)(35.6/1.27)^2} = 419.8 \text{ MPa but } \leq F_y$$

$$M_n = S_e F_y = 4,494 \text{ mm}^3 * 344.7 \text{ MPa} = 1,549 \text{ Nm Vertical loading}$$

$$5,073 \text{ mm}^3 * 344.7 \text{ MPa} = 1,748.7 \text{ Nm Horizontal load}$$

Use load factor of 1.6 and resistance factor of 1.0 (round shape using section modulus)

Determine allowable rail loads (ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights ( $M = wl^2/10$ )

$$\text{Vertical} \rightarrow \text{uniform} \rightarrow w = (1,549 \text{ Nm} * 10 / (1.6 * (1.5 \text{ m})^2)) = 4,302.8 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 1,549 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 3,227.1 \text{ N}$$

$$\text{Horizontal} \rightarrow \text{uniform} \rightarrow w = (1,748.7 \text{ Nm} * 10 / (1.6 * (1.5 \text{ m})^2)) = 4,857.5 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 1,748.7 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 3,643.1 \text{ N}$$



**GR 25 SERIES CAP RAIL For Brass:**

Alloy C26000, Cartridge Brass, 70% Cu, 30% Zn

Cap rail fabricated from cold rolled sheet

$$F_{yu} \geq 296.5 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 k \eta E_0}{12(1-\mu^2)(w/t)^2}$$

$$\eta = 0.49$$

$$k = 3(I_s/I_a)^{1/3} + 1 < 4.0 = 4.0 \text{ for circular shape}$$

$$\mu = 0.34$$

$$E_0 = 116.5 \times 10^3 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 * 4.0 * 0.49 * 116.5 \times 10^3 \text{ MPa}}{12(1-0.34^2)(35.6/1.27)^2} = 280.6 \text{ MPa (40.7 ksi) but } \leq F_y$$

$$S_{xx}: 4,494.0 \text{ mm}^3$$

$$S_{yy}: 5,073.0 \text{ mm}^3$$

$$M_n = S_e F_y = 4,494.0 \text{ mm}^3 * 280.6 \text{ MPa} = 1,261.0 \text{ Nm} \quad \text{Vertical loading}$$

$$5,073.0 \text{ mm}^3 * 280.6 \text{ MPa} = 1,423.5 \text{ Nm} \quad \text{Horizontal load}$$

Use load factor of 1.6 and resistance factor of 1.0 (round shape using section modulus)

Determine allowable rail loads (ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights ( $M = w l^2 / 10$ )

$$\text{Vertical} \rightarrow \text{uniform} \rightarrow w = (1,261 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 3,502.8 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 1,261 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 2,627.1 \text{ N}$$

$$\text{Horizontal} \rightarrow \text{uniform} \rightarrow w = (1,423.5 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 3,954.2 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 1,423.5 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 2,965.6 \text{ N}$$

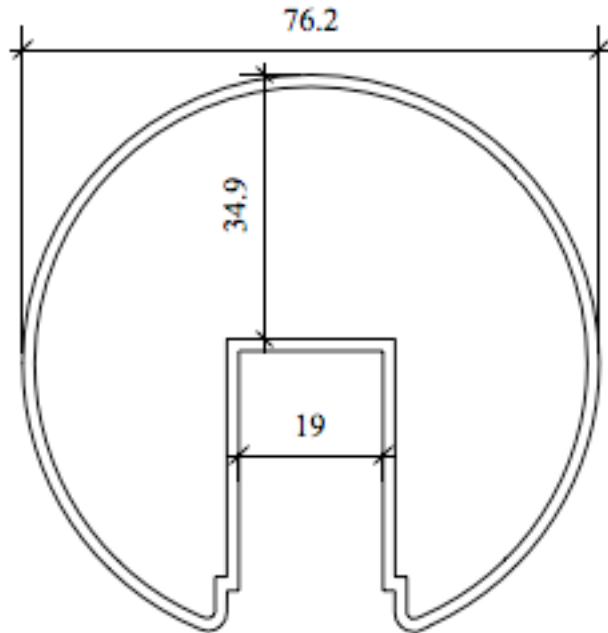
**Connector Sleeves****Corners**

The connector sleeves and corners are demonstrated as adequate based on strength for the 38mm size.

**CRL GR 30 SERIES CAP RAIL**

Used as the top rail on glass balustrade panel guardrails

Area: 487.36 mm<sup>2</sup>  
 $I_{xx}$ : 252,988 mm<sup>4</sup>  
 $I_{yy}$ : 271,897 mm<sup>4</sup>  
 $r_{xx}$ : 22.78mm  
 $r_{yy}$ : 23.62mm  
 $C_{xx}$ : 39.11mm  
 $C_{yy}$ : 38.1mm  
 $S_{xx}$ : 6,468.5 mm<sup>3</sup>  
 $S_{yy}$ : 7,134.4 mm<sup>3</sup>  
 $t$  = 1.27mm



Allowable stresses:

For stainless steel options:

design using SEI/ASCE 8-02

From Table A1, 344.7MPa for 1/4 hard

A304 stainless steel sheet used to form the rail.

$$F_{cr} = \frac{\pi^2 k \eta E_0}{12(1-\mu^2)(w/t)^2} \quad (\text{eq 3.3.1.1-9})$$

$\eta$  = 0.49 (from table A8a)

$k = 3(I_s/I_a)^{1/3} + 1 < 4.0 = 4.0$  for circular shape

$\mu$  = 0.3

$E_0$  = 186,200 MPa

$$F_{cr} = \frac{\pi^2 * 4.0 * 0.49 * 186,200 \text{ MPa}}{12(1-0.3^2)(35.6/1.27)^2} = 419.8 \text{ MPa but } \leq F_y$$

$M_n = S_e F_y = 6,468.5 \text{ mm}^3 * 344.7 \text{ MPa} = 2,229.7 \text{ Nm}$  Vertical loading

$7,134.4 \text{ mm}^3 * 344.7 \text{ MPa} = 2,459.2 \text{ Nm}$  Horizontal load

Use load factor of 1.6 and resistance factor of 1.0 (round shape using section modulus)

Determine allowable rail loads (ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights ( $M = wl^2/10$ )

Vertical  $\rightarrow$  uniform  $\rightarrow w = (2,229.7 \text{ Nm} * 10 / (1.6 * (1.5 \text{ m})^2)) = 6,193.6 \text{ N/m}$

concentrated  $\rightarrow P = 2,229.7 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 4,645.2 \text{ N}$

Horizontal  $\rightarrow$  uniform  $\rightarrow w = (2,459.2 \text{ Nm} * 10 / (1.6 * (1.5 \text{ m})^2)) = 6,831.1 \text{ N/m}$

concentrated  $\rightarrow P = 2,459.2 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 5,123.3 \text{ N}$

**GR 30 SERIES CAP RAIL For Brass:**

Alloy C26000, Cartridge Brass, 70% Cu, 30% Zn

Cap rail fabricated from cold rolled sheet

$$F_{yu} \geq 296.5 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 k \eta E_0}{12(1-\mu^2)(w/t)^2}$$

$$\eta = 0.49$$

$$k = 3(I_s/I_a)^{1/3} + 1 < 4.0 = 4.0 \text{ for circular shape}$$

$$\mu = 0.34$$

$$E_0 = 116.5 \times 10^3 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 * 4.0 * 0.49 * 116.5 \times 10^3 \text{ MPa}}{12(1-0.34^2)(35.6/1.27)^2} = 280.6 \text{ MPa (40.7 ksi) but } \leq F_y$$

$$S_{xx}: 6,468.5 \text{ mm}^3$$

$$S_{yy}: 7,134.4 \text{ mm}^3$$

$$M_n = S_e F_y = 6,468.5 \text{ mm}^3 * 280.6 \text{ MPa} = 1,815.1 \text{ Nm Vertical loading}$$

$$7,134.4 \text{ mm}^3 * 280.6 \text{ MPa} = 2,001.9 \text{ Nm Horizontal load}$$

Use load factor of 1.6 and resistance factor of 1.0 (round shape using section modulus)

Determine allowable rail loads (ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights ( $M = w l^2 / 10$ )

$$\text{Vertical} \rightarrow \text{uniform} \rightarrow w = (1,815.1 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 5,041.9 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 1,815.1 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 3,781.5 \text{ N}$$

$$\text{Horizontal} \rightarrow \text{uniform} \rightarrow \text{uniform} \rightarrow w = (2,001.9 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 5,560.8 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 2,001.9 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 4,170.7 \text{ N}$$

**Connector Sleeves****Corners**

The connector sleeves and corners are demonstrated as adequate based on strength for the 38mm size.

**CRL GR 35 SERIES CAP RAIL**

Used as the top rail on glass  
balustrade panel guardrails

Area: 558.5 mm<sup>2</sup>  
 $I_{xx}$ : 424,766 mm<sup>4</sup>  
 $I_{yy}$ : 437,638 mm<sup>4</sup>  
 $r_{xx}$ : 27.58mm  
 $r_{yy}$ : 27.99mm  
 $C_{xx}$ : 47.12mm  
 $C_{yy}$ : 44.45mm  
 $S_{xx}$ : 9,014.9 mm<sup>3</sup>  
 $S_{yy}$ : 9,845.6 mm<sup>3</sup>  
 $t = 1.27$ mm

Allowable stresses:

For stainless steel options:  
 design using SEI/ASCE 8-02  
 From Table A1, 344.7MPa for 1/4  
 hard A304 stainless steel sheet  
 used to form the rail.

$$F_{cr} = \frac{\pi^2 k \eta E_0}{12(1-\mu^2)(w/t)^2} \quad (\text{eq 3.3.1.1-9})$$

$$12(1-\mu^2)(w/t)^2$$

$$\eta = 0.49 \text{ (from table A8a)}$$

$$k = 3(I_s/I_a)^{1/3} + 1 < 4.0 = 4.0 \text{ for circular shape}$$

$$\mu = 0.3$$

$$E_0 = 186,200 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 * 4.0 * 0.49 * 186,200 \text{ MPa}}{12(1-0.3^2)(35.6/1.27)^2} = 419.8 \text{ MPa but } \leq F_y$$

$$M_n = S_e F_y = 9,014.9 \text{ mm}^3 * 344.7 \text{ MPa} = 3,107.4 \text{ Nm Vertical loading}$$

$$9,845.6 \text{ mm}^3 * 344.7 \text{ MPa} = 3,393.8 \text{ Nm Horizontal load}$$

Use load factor of 1.6 and resistance factor of 1.0 (round shape using section modulus)

Determine allowable rail loads (ignoring deflection) for a maximum span of 1.5m

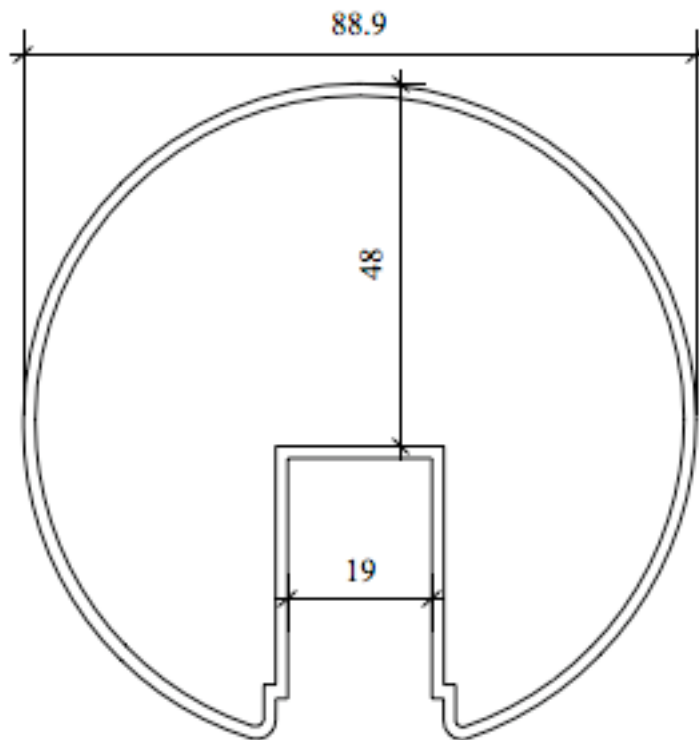
Rail is continuous over multiple glass lights ( $M = w l^2 / 10$ )

$$\text{Vertical } \rightarrow \text{ uniform } \rightarrow w = (3,107.4 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 8,631.7 \text{ N/m}$$

$$\text{concentrated } \rightarrow P = 3,107.4 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 6,473.7 \text{ N}$$

$$\text{Horizontal } \rightarrow \text{ uniform } \rightarrow w = (3,393.8 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 9,427.2 \text{ N/m}$$

$$\text{concentrated } \rightarrow P = 3,393.8 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 7,070.4 \text{ N}$$



**GR 35 SERIES CAP RAIL For Brass:**

Alloy C26000, Cartridge Brass, 70% Cu, 30% Zn

Cap rail fabricated from cold rolled sheet

$$F_{yu} \geq 296.5 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 k \eta E_0}{12(1-\mu^2)(w/t)^2}$$

$$\eta = 0.49$$

$$k = 3(I_s/I_a)^{1/3} + 1 < 4.0 = 4.0 \text{ for circular shape}$$

$$\mu = 0.34$$

$$E_0 = 116.5 \times 10^3 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 * 4.0 * 0.49 * 116.5 \times 10^3 \text{ MPa}}{12(1-0.34^2)(35.6/1.27)^2} = 280.6 \text{ MPa (40.7 ksi) but } \leq F_y$$

$$S_{xx}: 9,014.9 \text{ mm}^3$$

$$S_{yy}: 9,845.6 \text{ mm}^3$$

$$M_n = S_e F_y = 9,014.9 \text{ mm}^3 * 280.6 \text{ MPa} = 2,529.6 \text{ Nm} \quad \text{Vertical loading}$$

$$9,845.6 \text{ mm}^3 * 280.6 \text{ MPa} = 2,762.7 \text{ Nm} \quad \text{Horizontal load}$$

Use load factor of 1.6 and resistance factor of 1.0 (round shape using section modulus)

Determine allowable rail loads (ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights ( $M = w l^2 / 10$ )

$$\text{Vertical} \rightarrow \text{uniform} \rightarrow w = (2,529.6 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 7,026.7 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 2,762.7 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 5,755.6 \text{ N}$$

$$\text{Horizontal} \rightarrow \text{uniform} \rightarrow \text{uniform} \rightarrow w = (2,762.7 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 7,674.2 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 2,762.7 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 5,755.6 \text{ N}$$

**Connector Sleeves****Corners**

The connector sleeves and corners are demonstrated as adequate based on strength for the 38mm size.

**CRL GR 40 SERIES CAP RAIL**

Used as the top rail on glass balustrade panel guardrails

Area: 614.3 mm<sup>2</sup>

I<sub>xx</sub>: 646,553 mm<sup>4</sup>

I<sub>yy</sub>: 636,584 mm<sup>4</sup>

r<sub>xx</sub>: 32.44mm

r<sub>yy</sub>: 32.19mm

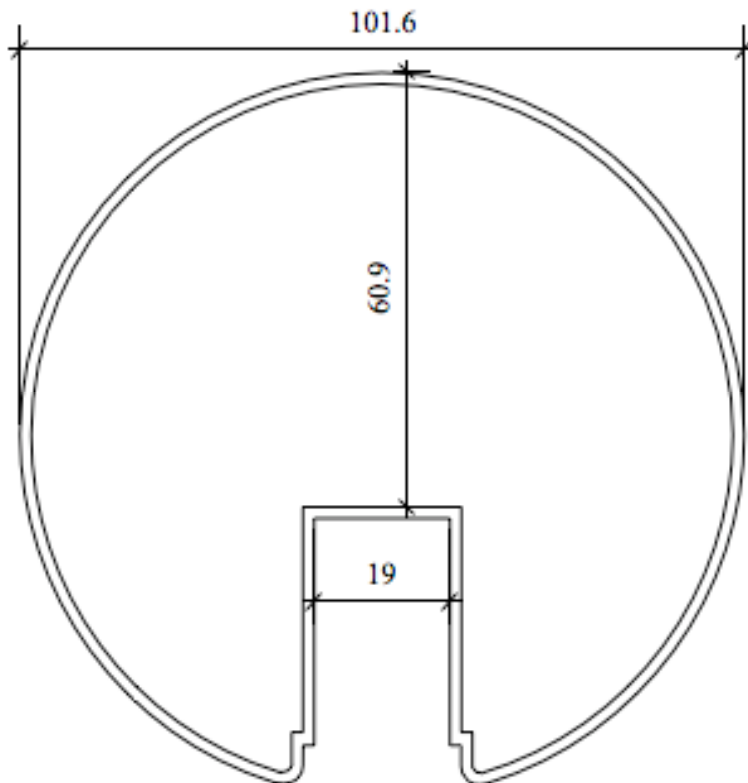
C<sub>xx</sub>: 54.13mm

C<sub>yy</sub>: 50.80mm

S<sub>xx</sub>: 11,944.8 mm<sup>3</sup>

S<sub>yy</sub>: 12,529.6 mm<sup>3</sup>

t = 1.27mm



Allowable stresses:

For stainless steel  
options: design using  
SEI/ASCE 8-02  
From Table A1, 344.7MPa  
for 1/4 hard A304 stainless  
steel sheet used to form the  
rail.

$$F_{cr} = \frac{\pi^2 k \eta E_0}{12(1-\mu^2)(w/t)^2}$$

(eq 3.3.1.1-9)

$$12(1-\mu^2)(w/t)^2$$

$\eta = 0.49$  (from table A8a)

$k = 3(I_s/I_a)^{1/3} + 1 < 4.0 = 4.0$  for circular shape

$\mu = 0.3$

$E_0 = 186,200$  MPa

$$F_{cr} = \frac{\pi^2 * 4.0 * 0.49 * 186,200 \text{ MPa}}{12(1-0.3^2)(35.6/1.27)^2} = 419.8 \text{ MPa but } \leq F_y$$

$M_n = S_e F_y = 11,944.8 \text{ mm}^3 * 344.7 \text{ MPa} = 4,117.4 \text{ Nm}$  Vertical loading

$12,529.6 \text{ mm}^3 * 344.7 \text{ MPa} = 4,319.0 \text{ Nm}$  Horizontal load

Use load factor of 1.6 and resistance factor of 1.0 (round shape using section modulus)

Determine allowable rail loads (ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights ( $M = w l^2 / 10$ )

Vertical  $\rightarrow$  uniform  $\rightarrow w = (4,117.4 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 11,437.2 \text{ N/m}$

concentrated  $\rightarrow P = 4,117.4 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 8,577.9 \text{ N}$

Horizontal  $\rightarrow$  uniform  $\rightarrow w = (4,319.0 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 11,997.2 \text{ N/m}$

concentrated  $\rightarrow P = 4,319.0 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 8,997.9 \text{ N}$

**GR 40 SERIES CAP RAIL For Brass:**

Alloy C26000, Cartridge Brass, 70% Cu, 30% Zn

Cap rail fabricated from cold rolled sheet

$$F_{yu} \geq 296.5 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 k \eta E_0}{12(1-\mu^2)(w/t)^2}$$

$$\eta = 0.49$$

$$k = 3(I_s/I_a)^{1/3} + 1 < 4.0 = 4.0 \text{ for circular shape}$$

$$\mu = 0.34$$

$$E_0 = 116.5 \times 10^3 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 * 4.0 * 0.49 * 116.5 \times 10^3 \text{ MPa}}{12(1-0.34^2)(35.6/1.27)^2} = 280.6 \text{ MPa (40.7 ksi) but } \leq F_y$$

$$S_{xx}: 11,944.8 \text{ mm}^3$$

$$S_{yy}: 12,529.6 \text{ mm}^3$$

$$M_n = S_e F_y = 11,944.8 \text{ mm}^3 * 280.6 \text{ MPa} = 3,351.7 \text{ Nm} \quad \text{Vertical loading}$$

$$12,529.6 \text{ mm}^3 * 280.6 \text{ MPa} = 3,515.8 \text{ Nm} \quad \text{Horizontal load}$$

Use load factor of 1.6 and resistance factor of 1.0 (round shape using section modulus)

Determine allowable rail loads (ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights ( $M = w l^2 / 10$ )

$$\text{Vertical} \rightarrow \text{uniform} \rightarrow w = (3,351.7 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 9,310.3 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 3,351.7 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 6,982.7 \text{ N}$$

$$\text{Horizontal} \rightarrow \text{uniform} \rightarrow w = (3,515.8 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 9,766.1 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 3,515.8 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 7,324.6 \text{ N}$$

**Connector Sleeves****Corners**

The connector sleeves and corners are demonstrated as adequate based on strength for the 38mm size.

**CRL GR 207 SERIES CAP RAIL**

Used as the top rail on glass balustrade panel guardrails.

Use with 19mm glass balustrades

Area: 341.28 mm<sup>2</sup>I<sub>xx</sub>: 58,557 mm<sup>4</sup>I<sub>yy</sub>: 92,312 mm<sup>4</sup>r<sub>xx</sub>: 13.10mmr<sub>yy</sub>: 16.45mmC<sub>xx</sub>: 23.59mmC<sub>yy</sub>: 25.40mmS<sub>xx</sub>: 2,482.7 mm<sup>3</sup>S<sub>yy</sub>: 3,634.3 mm<sup>3</sup>

t = 1.27mm

Allowable stresses:

For stainless steel options: design using SEI/ASCE 8-02

From Table A1, F<sub>y</sub> = 344.7MPa for 1/4 hard 304 stainless steel cold formed sheet

$$F_{cr} = \frac{\pi^2 k \eta E_0}{12(1-\mu^2)(w/t)^2} \quad (\text{eq 3.3.1.1-9})$$

η = 0.49 (from table A8a)

k = 3(I<sub>s</sub>/I<sub>a</sub>)<sup>1/3</sup> + 1 < 4.0 = 4.0 for circular shape

μ = 0.3

E<sub>0</sub> = 186,200 MPa

$$F_{cr} = \frac{\pi^2 * 4.0 * 0.49 * 186,200 \text{MPa}}{12(1-0.3^2)(35.6/1.27)^2} = 419.8 \text{MPa but } \leq F_y$$

M<sub>n</sub> = S<sub>e</sub>F<sub>y</sub> = 2,482.7 mm<sup>3</sup>\*344.7MPa = 855.8Nm Vertical loading3,634.3 mm<sup>3</sup>\*344.7MPa = 1,252.7Nm Horizontal load

Use load factor of 1.6 and resistance factor of 1.0 (round shape using section modulus)

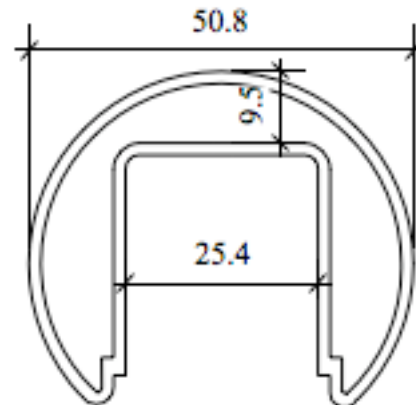
Determine allowable rail loads (ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights (M = wl<sup>2</sup>/10)Vertical → uniform → w = (855.8Nm • 10 / (1.6\*(1.5m)<sup>2</sup>)) = 2,377.2N/m

concentrated → P = 855.8Nm\*5 / (1.6\*1.5m) = 1,782.9N

Horizontal → uniform → w = (1,252.7Nm • 10 / (1.6\*(1.5m)<sup>2</sup>)) = 3,479.7 N/m

concentrated → P = 1,252.7Nm\*5 / (1.6\*1.5m) = 2,609.8N





GR 207 SERIES CAP RAIL For Brass:

Alloy C26000, Cartridge Brass, 70% Cu, 30% Zn

Cap rail fabricated from cold rolled sheet

$$F_{yu} \geq 296.5 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 k \eta E_0}{12(1-\mu^2)(w/t)^2}$$

$$\eta = 0.49$$

$$k = 3(I_s/I_a)^{1/3} + 1 < 4.0 = 4.0 \text{ for circular shape}$$

$$\mu = 0.34$$

$$E_0 = 116.5 \times 10^3 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 * 4.0 * 0.49 * 116.5 \times 10^3 \text{ MPa}}{12(1-0.34^2)(35.6/1.27)^2} = 280.6 \text{ MPa (40.7 ksi) but } \leq F_y$$

$$S_{xx}: 2,482.7 \text{ mm}^3$$

$$S_{yy}: 3,634.3 \text{ mm}^3$$

$$M_n = S_e F_y = 2,482.7 \text{ mm}^3 * 280.6 \text{ MPa} = 696.6 \text{ Nm} \quad \text{Vertical loading}$$

$$3,634.3 \text{ mm}^3 * 280.6 \text{ MPa} = 1,019.8 \text{ Nm} \quad \text{Horizontal load}$$

Use load factor of 1.6 and resistance factor of 1.0 (round shape using section modulus)

Determine allowable rail loads (ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights ( $M = wl^2/10$ )

$$\text{Vertical } \rightarrow \text{uniform } \rightarrow w = (696.6 \text{ Nm} * 10 / (1.6 * (1.5 \text{ m})^2)) = 1,935 \text{ N/m}$$

$$\text{concentrated } \rightarrow P = 696.6 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 1,451.3 \text{ N}$$

$$\text{Horizontal } \rightarrow \text{uniform } \rightarrow w = (1,019.8 \text{ Nm} * 10 / (1.6 * (1.5 \text{ m})^2)) = 2,832.8 \text{ N/m}$$

$$\text{concentrated } \rightarrow P = 1,019.8 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 2,124.6 \text{ N}$$

Connector Sleeves

### Corners

The connector sleeves and corners are demonstrated as adequate based on strength for the 38mm size.

**CRL GR 257 SERIES CAP RAIL**

Used as the top rail on glass balustrade panel guardrails

Use with 19mm glass balustrades

Area: 409.2 mm<sup>2</sup>

I<sub>xx</sub>: 122,820 mm<sup>4</sup>

I<sub>yy</sub>: 167,126 mm<sup>4</sup>

r<sub>xx</sub>: 17.33mm

r<sub>yy</sub>: 20.21mm

C<sub>xx</sub>: 29.58mm

C<sub>yy</sub>: 31.75mm

S<sub>xx</sub>: 4,152.2 mm<sup>3</sup>

S<sub>yy</sub>: 5,263.8 mm<sup>3</sup>

t = 1.27mm

Allowable stresses:

For stainless steel options: design using SEI/ASCE 8-02

From Table A1, F<sub>y</sub> = 344.7MPa for 1/4 hard 304 stainless steel cold formed sheet

$$F_{cr} = \frac{\pi^2 k \eta E_0}{12(1-\mu^2)(w/t)^2} \quad (\text{eq 3.3.1.1-9})$$

$\eta = 0.49$  (from table A8a)

$k = 3(I_s/I_a)^{1/3} + 1 < 4.0 = 4.0$  for circular shape

$\mu = 0.3$

E<sub>0</sub> = 186,200 MPa

$$F_{cr} = \frac{\pi^2 * 4.0 * 0.49 * 186,200 \text{MPa}}{12(1-0.3^2)(35.6/1.27)^2} = 419.8 \text{MPa but } \leq F_y$$

M<sub>n</sub> = S<sub>e</sub>F<sub>y</sub> = 4,152.2 mm<sup>3</sup>\*344.7MPa = 1,431.3Nm Vertical loading

5,263.8 mm<sup>3</sup>\*344.7MPa = 1,814.4Nm Horizontal load

Use load factor of 1.6 and resistance factor of 1.0 (round shape using section modulus)

Determine allowable rail loads(ignoring deflection) for a maximum span of 1.5m

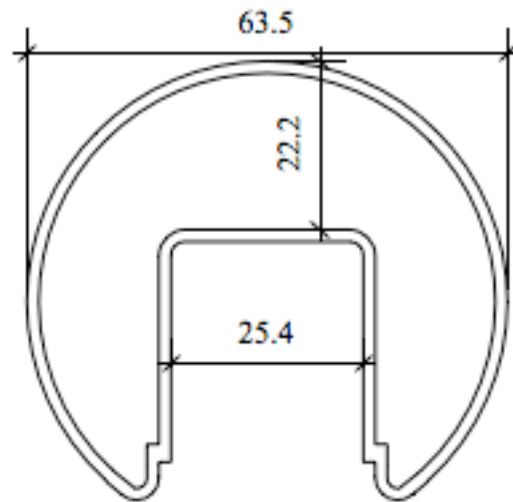
Rail is continuous over multiple glass lights (M = w<sup>2</sup>/10)

Vertical → uniform → w = (1,431.3Nm • 10 / (1.6 \* (1.5m)<sup>2</sup>)) = 3,975.8N/m

concentrated → P = 1,431.3Nm \* 5 / (1.6 \* 1.5m) = 2,981.9N

Horizontal → uniform → w = (1,814.4Nm • 10 / (1.6 \* (1.5m)<sup>2</sup>)) = 5,040 N/m

concentrated → P = 1,814.4Nm \* 5 / (1.6 \* 1.5m) = 3,780N



GR 257 SERIES CAP RAIL For Brass:

Alloy C26000, Cartridge Brass, 70% Cu, 30% Zn

Cap rail fabricated from cold rolled sheet

$$F_{yu} \geq 296.5 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 k \eta E_0}{12(1-\mu^2)(w/t)^2}$$

$$\eta = 0.49$$

$$k = 3(I_s/I_a)^{1/3} + 1 < 4.0 = 4.0 \text{ for circular shape}$$

$$\mu = 0.34$$

$$E_0 = 116.5 \times 10^3 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 * 4.0 * 0.49 * 116.5 \times 10^3 \text{ MPa}}{12(1-0.34^2)(35.6/1.27)^2} = 280.6 \text{ MPa (40.7 ksi) but } \leq F_y$$

$$S_{xx}: 4,152.2 \text{ mm}^3$$

$$S_{yy}: 5,263.8 \text{ mm}^3$$

$$M_n = S_e F_y = 4,152.2 \text{ mm}^3 * 280.6 \text{ MPa} = 1,165.1 \text{ Nm} \quad \text{Vertical loading}$$

$$5,263.8 \text{ mm}^3 * 280.6 \text{ MPa} = 1,477.0 \text{ Nm} \quad \text{Horizontal load}$$

Use load factor of 1.6 and resistance factor of 1.0 (round shape using section modulus)

Determine allowable rail loads (ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights ( $M = wl^2/10$ )

$$\text{Vertical} \rightarrow \text{uniform} \rightarrow w = (1,165.1 \text{ Nm} * 10 / (1.6 * (1.5 \text{ m})^2)) = 3,236.4 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 1,165.1 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 2,427.3 \text{ N}$$

$$\text{Horizontal} \rightarrow \text{uniform} \rightarrow w = (1,477.0 \text{ Nm} * 10 / (1.6 * (1.5 \text{ m})^2)) = 4,102.8 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 1,477.0 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 3,077.1 \text{ N}$$

Connector Sleeves

### Corners

The connector sleeves and corners are demonstrated as adequate based on strength for the 38mm size.

**CRL GR 307 SERIES CAP RAIL**

Used as the top rail on glass balustrade panel guardrails.

Use with 19mm glass balustrades

Area: 479.3 mm<sup>2</sup>

I<sub>xx</sub>: 233,090 mm<sup>4</sup>

I<sub>yy</sub>: 281,765 mm<sup>4</sup>

r<sub>xx</sub>: 22.05mm

r<sub>yy</sub>: 24.25mm

C<sub>xx</sub>: 37.94mm

C<sub>yy</sub>: 38.1mm

S<sub>xx</sub>: 6,143.3 mm<sup>3</sup>

S<sub>yy</sub>: 7,360.2 mm<sup>3</sup>

t = 1.27mm

Allowable stresses:

For stainless steel options: design using SEI/ASCE 8-02

From Table A1, F<sub>y</sub> = 344.7MPa for 1/4 hard 304 stainless steel cold formed sheet

$$F_{cr} = \frac{\pi^2 k \eta E_0}{12(1-\mu^2)(w/t)^2} \quad (\text{eq 3.3.1.1-9})$$

$\eta = 0.49$  (from table A8a)

$k = 3(I_s/I_a)^{1/3} + 1 < 4.0 = 4.0$  for circular shape

$\mu = 0.3$

E<sub>0</sub> = 186,200 MPa

$$F_{cr} = \frac{\pi^2 * 4.0 * 0.49 * 186,200 \text{MPa}}{12(1-0.3^2)(35.6/1.27)^2} = 419.8 \text{MPa but } \leq F_y$$

M<sub>n</sub> = S<sub>e</sub>F<sub>y</sub> = 6,143.3 mm<sup>3</sup>\*344.7MPa = 2,117.6Nm Vertical loading

7,360.2 mm<sup>3</sup>\*344.7MPa = 2,537.1Nm Horizontal load

Use load factor of 1.6 and resistance factor of 1.0 (round shape using section modulus)

Determine allowable rail loads(ignoring deflection) for a maximum span of 1.5m

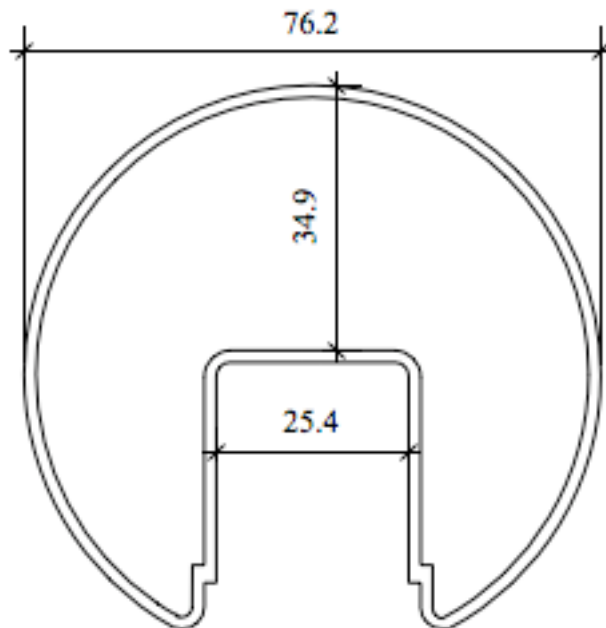
Rail is continuous over multiple glass lights (M = w<sup>2</sup>/10) or Pl/5

Vertical → uniform → w = (2,117.6Nm • 10 / (1.6 \* (1.5m)<sup>2</sup>)) = 5,882.2N/m

concentrated → P = 2,117.6Nm \* 5 / (1.6 \* 1.5m) = 4,411.7N

Horizontal → uniform → w = (2,537.1Nm • 10 / (1.6 \* (1.5m)<sup>2</sup>)) = 7,047.5N/m

concentrated → P = 2,537.1Nm \* 5 / (1.6 \* 1.5m) = 5,285.6N



GR 307 SERIES CAP RAIL For Brass:

Alloy C26000, Cartridge Brass, 70% Cu, 30% Zn

Cap rail fabricated from cold rolled sheet

$$F_{yu} \geq 296.5 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 k \eta E_0}{12(1-\mu^2)(w/t)^2}$$

$$\eta = 0.49$$

$$k = 3(I_s/I_a)^{1/3} + 1 < 4.0 = 4.0 \text{ for circular shape}$$

$$\mu = 0.34$$

$$E_0 = 116.5 \times 10^3 \text{ MPa}$$

$$F_{cr} = \frac{\pi^2 * 4.0 * 0.49 * 116.5 \times 10^3 \text{ MPa}}{12(1-0.34^2)(35.6/1.27)^2} = 280.6 \text{ MPa (40.7 ksi) but } \leq F_y$$

$$S_{xx}: 6,143.3 \text{ mm}^3$$

$$S_{yy}: 7,360.2 \text{ mm}^3$$

$$M_n = S_e F_y = 6,143.3 \text{ mm}^3 * 280.6 \text{ MPa} = 1,723.8 \text{ Nm} \quad \text{Vertical loading}$$

$$7,360.2 \text{ mm}^3 * 280.6 \text{ MPa} = 2,065.3 \text{ Nm} \quad \text{Horizontal load}$$

Use load factor of 1.6 and resistance factor of 1.0 (round shape using section modulus)

Determine allowable rail loads (ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights ( $M = wl^2/10$ )

$$\text{Vertical} \rightarrow \text{uniform} \rightarrow w = (1,723.8 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 4,788.3 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 1,723.8 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 3,591.2 \text{ N}$$

$$\text{Horizontal} \rightarrow \text{uniform} \rightarrow w = (2,065.3 \text{ Nm} \cdot 10 / (1.6 * (1.5 \text{ m})^2)) = 5,736.9 \text{ N/m}$$

$$\text{concentrated} \rightarrow P = 2,065.3 \text{ Nm} * 5 / (1.6 * 1.5 \text{ m}) = 4,302.7 \text{ N}$$

Connector Sleeves

### Corners

The connector sleeves and corners are demonstrated as adequate based on strength for the 38mm size.

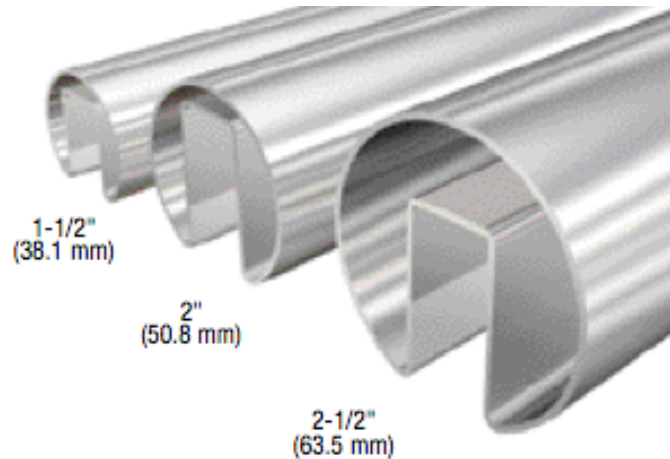
**Roll Formed Cap Rails:**

304 Stainless steel

GRRF15 38mm rail is structurally equivalent to GR15 38mm rail.

GRRF20 51mm rail is structurally equivalent to GR20 51mm rail.

GRRF25 63mm rail is structurally equivalent to GR25 63mm rail.



**Custom Angle Corners**

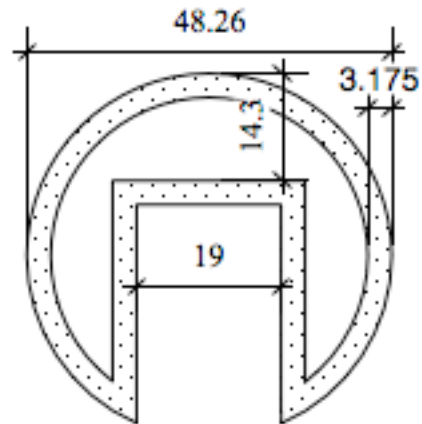
Corners may be welded at any angle, vertical or horizontal angles.

Compound angles may be used.

The strength of the angle is not decreased below that for the 90° angle used for the standard calculation therefore strength adequacy is demonstrated for all angles.

**ALUMINUM CAP RAILS****GR19 Aluminum**Area: 623.2 mm<sup>2</sup>I<sub>xx</sub>: 100,768 mm<sup>4</sup>I<sub>yy</sub>: 135,542 mm<sup>4</sup>r<sub>xx</sub>: 12.72mmr<sub>yy</sub>: 14.80mmC<sub>xx</sub>: 24.08mmC<sub>yy</sub>: 24.13mmS<sub>xx</sub>: 4,184.6 mm<sup>3</sup> topS<sub>xx</sub>: 4,167.4 mm<sup>3</sup> bottomS<sub>yy</sub>: 5,658.6 mm<sup>3</sup>

t = 3.175mm

Allowable stresses **ADM Table 2-24 6063-T6 Aluminum**

$$F_{Cb} \rightarrow R_b/t = \frac{20.95\text{mm}}{3.175\text{mm}} = 6.6 \text{ line 16.1}$$

$$F_{Cb} = 127.55\text{MPa} - 4.10(6.6)^{1/2} = 117.0\text{MPa}$$

$$M_{\text{all horiz}} = 117.0\text{MPa} \cdot (5,658.6 \text{ mm}^3) = 662.1\text{Nm}$$

For vertical load → bottom in tension top comp.

$$F_b = 124.1\text{MPa}$$

$$\text{bottom stress: } M_{\text{all vert}} = (4,167.4 \text{ mm}^3) \cdot 124.1\text{MPa} = 517.2\text{Nm} \text{ or}$$

$$\text{top stress: } = (4,184.6 \text{ mm}^3) \cdot 117.0\text{MPa} = 489.6\text{Nm} \text{ controls}$$

Determine allowable rail loads(ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights ( $M = wL^2/10$ ) or  $Pl/5$ 

$$\text{Vertical} \rightarrow \text{uniform} \rightarrow w = (489.6\text{Nm} \cdot 10 / (1.6 \cdot (1.5\text{m})^2)) = 1,360\text{N/m}$$

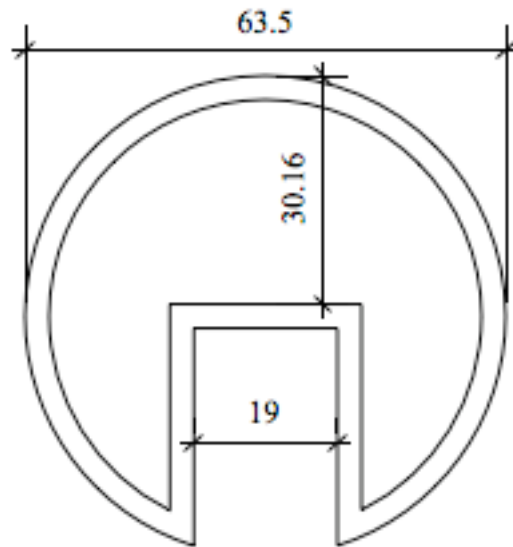
$$\text{concentrated} \rightarrow P = 489.6\text{Nm} \cdot 5 / (1.6 \cdot 1.5\text{m}) = 1,020\text{N}$$

$$\text{Horizontal} \rightarrow \text{uniform} \rightarrow w = (662.1\text{Nm} \cdot 10 / (1.6 \cdot (1.5\text{m})^2)) = 1,753.6\text{N/m}$$

$$\text{concentrated} \rightarrow P = 662.1\text{Nm} \cdot 5 / (1.6 \cdot 1.5\text{m}) = 1,379.4\text{N}$$

**GR25 Aluminum**Area: 777.85 mm<sup>2</sup>I<sub>xx</sub>: 259,022 mm<sup>4</sup>I<sub>yy</sub>: 296,181 mm<sup>4</sup>r<sub>xx</sub>: 18.25mmr<sub>yy</sub>: 19.51mmC<sub>xx</sub>: 32.23mmC<sub>yy</sub>: 31.75mmS<sub>xx</sub>: 8,283.4 mm<sup>3</sup> topS<sub>xx</sub>: 8,035.7 mm<sup>3</sup> bottomS<sub>yy</sub>: 9,328.5 mm<sup>3</sup>

t = 3.175mm

Allowable stresses **ADM Table 2-24****6063-T6 Aluminum**

$$F_{Cb} \rightarrow R_b/t = \frac{28.575\text{mm}}{3.175\text{mm}} = 9 \text{ line 16.1}$$

$$F_{Cb} = 127.55\text{MPa} - 4.10(9)^{1/2} = 115.25\text{MPa}$$

$$M_{\text{all horiz}} = 115.25\text{MPa} \cdot (9,328.5\text{mm}^3) = 1,075.1\text{Nm}$$

For vertical load  $\rightarrow$  bottom in tension top comp.

$$F_b = 124.1\text{MPa}$$

$$\text{bottom stress: } M_{\text{all vert}} = (8,035.7\text{mm}^3) \cdot 124.1\text{MPa} = 997.2\text{Nm} \text{ or}$$

$$\text{top stress: } = (8,283.4\text{mm}^3) \cdot 115.25\text{MPa} = 954.7\text{Nm} \text{ controls}$$

Determine allowable rail loads(ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights ( $M = wl^2/10$ ) or  $Pl/5$ 

$$\text{Vertical } \rightarrow \text{uniform } \rightarrow w = (954.7\text{Nm} \cdot 10 / (1.6 \cdot (1.5\text{m})^2)) = 2,651.9\text{N/m}$$

$$\text{concentrated } \rightarrow P = 954.7\text{Nm} \cdot 5 / (1.6 \cdot 1.5\text{m}) = 1,989\text{N}$$

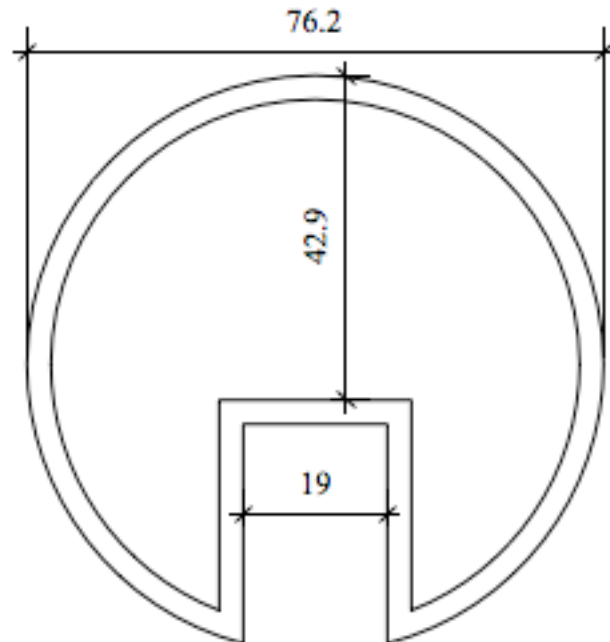
$$\text{Horizontal } \rightarrow \text{uniform } \rightarrow w = (1,075.1\text{Nm} \cdot 10 / (1.6 \cdot (1.5\text{m})^2)) = 2,986.1\text{N/m}$$

$$\text{concentrated } \rightarrow P = 1,075.1\text{Nm} \cdot 5 / (1.6 \cdot 1.5\text{m}) = 2,239.8\text{N}$$



**GR30 Aluminum**Area: 907.5 mm<sup>2</sup>I<sub>xx</sub>: 482,853 mm<sup>4</sup>I<sub>yy</sub>: 508,642 mm<sup>4</sup>r<sub>xx</sub>: 23.07mmr<sub>yy</sub>: 23.67mmC<sub>xx</sub>: 39.84mmC<sub>yy</sub>: 38.10mmS<sub>xx</sub>: 13,279.8 mm<sup>3</sup> topS<sub>xx</sub>: 12,119.0 mm<sup>3</sup> bottomS<sub>yy</sub>: 13,350.2 mm<sup>3</sup>

t = 3.175mm

Allowable stresses **ADM Table 2-24****6063-T6 Aluminum**

$$F_{Cb} \rightarrow R_b/t = \frac{36.51\text{mm}}{3.175\text{mm}} = 11.5 \text{ line 16.1}$$

$$F_{Cb} = 127.55\text{MPa} - 4.10(11.5)^{1/2} = 113.65\text{MPa}$$

$$M_{\text{all horiz}} = 113.65\text{MPa} \cdot (13,350.2\text{mm}^3) = 1,517.3\text{Nm}$$

For vertical load → bottom in tension top comp.

$$F_b = 124.1\text{MPa}$$

$$\text{bottom stress: } M_{\text{all vert}} = (12,119.0\text{mm}^3) \cdot 124.1\text{MPa} = 1,504.0\text{Nm} \text{ or}$$

$$\text{top stress: } = (13,279.8 \text{ mm}^3) \cdot 113.65\text{MPa} = 1,509.2\text{Nm} \text{ controls}$$

Determine allowable rail loads(ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights ( $M = wl^2/10$ ) or  $Pl/5$ 

$$\text{Vertical} \rightarrow \text{uniform} \rightarrow w = (1,504.0\text{Nm} \cdot 10 / (1.6 \cdot (1.5\text{m})^2)) = 4,177.8\text{N/m}$$

$$\text{concentrated} \rightarrow P = 1,504.0\text{Nm} \cdot 5 / (1.6 \cdot 1.5\text{m}) = 3,133\text{N}$$

$$\text{Horizontal} \rightarrow \text{uniform} \rightarrow w = (1,517.3\text{Nm} \cdot 10 / (1.6 \cdot (1.5\text{m})^2)) = 4,215.1\text{N/m}$$

$$\text{concentrated} \rightarrow P = 1,517.3\text{Nm} \cdot 5 / (1.6 \cdot 1.5\text{m}) = 3,161\text{N}$$

**GR35 Aluminum**Area: 1,036.2 mm<sup>2</sup>I<sub>xx</sub>: 808,339 mm<sup>4</sup>I<sub>yy</sub>: 808,918 mm<sup>4</sup>r<sub>xx</sub>: 27.93mmr<sub>yy</sub>: 27.94mmC<sub>xx</sub>: 47.15mmC<sub>yy</sub>: 44.45mmS<sub>xx</sub>: 19,361.4 mm<sup>3</sup> topS<sub>xx</sub>: 17,144.1 mm<sup>3</sup> bottomS<sub>yy</sub>: 18,198.4 mm<sup>3</sup>

t = 3.175mm

Allowable stresses **ADM****Table 2-24 6063-T6****Aluminum**

line 16.1

$$F_{Cb} \rightarrow R_b/t = \frac{42.86\text{mm}}{3.175\text{mm}} = 13.5$$

$$F_{Cb} = 127.55\text{MPa} - 4.10(13.5)^{1/2} \\ = 112.49\text{MPa}$$

$$M_{\text{all horiz}} = 112.49\text{MPa} \cdot (18,198.4\text{mm}^3) = 2,047.1\text{Nm}$$

For vertical load → bottom in tension top comp.

$$F_b = 124.1\text{MPa}$$

$$\text{bottom stress: } M_{\text{all vert}} = (17,144.1\text{mm}^3) \cdot 124.1\text{MPa} = 2,127.6\text{Nm} \text{ or}$$

$$\text{top stress: } = (19,361.4\text{mm}^3) \cdot 112.49\text{MPa} = 2,178.0\text{Nm} \text{ controls}$$

Determine allowable rail loads(ignoring deflection) for a maximum span of 1.5m

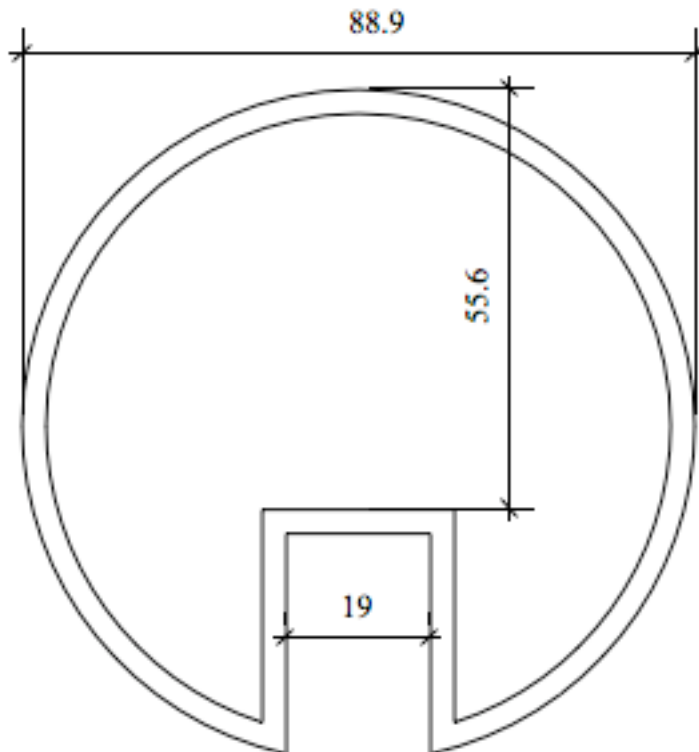
Rail is continuous over multiple glass lights ( $M = wL^2/10$ ) or  $PL/5$ 

$$\text{Vertical} \rightarrow \text{uniform} \rightarrow w = (2,127.6\text{Nm} \cdot 10 / (1.6 \cdot (1.5\text{m})^2)) = 5,910\text{N/m}$$

$$\text{concentrated} \rightarrow P = 2,127.6\text{Nm} \cdot 5 / (1.6 \cdot 1.5\text{m}) = 4,432.5\text{N}$$

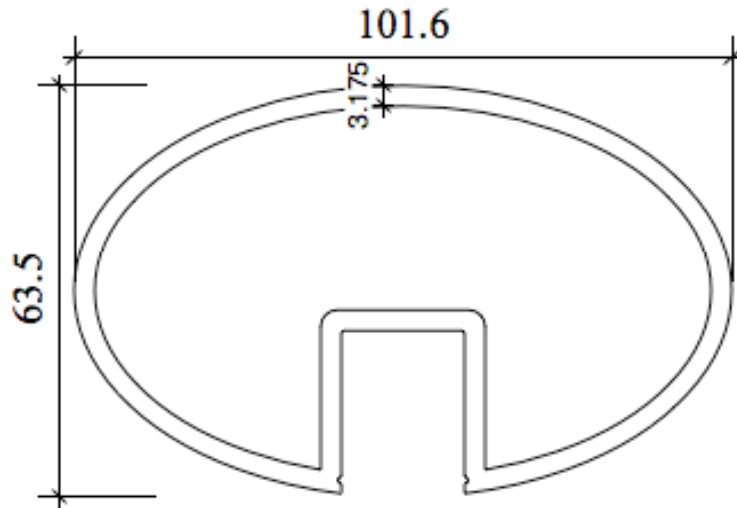
$$\text{Horizontal} \rightarrow \text{uniform} \rightarrow w = (2,047.1\text{Nm} \cdot 10 / (1.6 \cdot (1.5\text{m})^2)) = 5,686.4\text{N/m}$$

$$\text{concentrated} \rightarrow P = 2,047.1\text{Nm} \cdot 5 / (1.6 \cdot 1.5\text{m}) = 4,264.8\text{N}$$



**GROV4Aluminum**Area: 946.1 mm<sup>2</sup>I<sub>xx</sub>: 395,502 mm<sup>4</sup>I<sub>yy</sub>: 864.942 mm<sup>4</sup>r<sub>xx</sub>: 20.45mmr<sub>yy</sub>: 30.24mmC<sub>xx</sub>: 32.65mmC<sub>yy</sub>: 50.80mmS<sub>xx</sub>: 12,820.2 mm<sup>3</sup> topS<sub>xx</sub>: 12,112.6 mm<sup>3</sup> bottomS<sub>yy</sub>: 17,026.1 mm<sup>3</sup>

t = 3.175mm

Allowable stresses **ADM****Table 2-24 6063-T6****Aluminum**

line 16.1

$$F_{Cb} \rightarrow R_b/t = \frac{49.21\text{mm}}{3.175\text{mm}} = 15.5 \quad \text{top of rail}$$

$$F_{Cb} = 127.55\text{MPa} - 4.10(15.5)^{1/2} = 111.41\text{MPa} \quad \text{Compression on top of rail}$$

$$F_{Cb} \rightarrow R_b/t = \frac{17.41\text{mm}}{3.175\text{mm}} = 5.48 \quad \text{top of rail}$$

$$F_{Cb} = 127.55\text{MPa} - 4.10(5.48)^{1/2} = 117.95\text{MPa} \quad \text{Compression on top of rail}$$

$$M_{\text{all horiz}} = 117.95\text{MPa} \cdot (17,026.1\text{mm}^3) = 2,008.2\text{Nm}$$

For vertical load → bottom in tension top comp.

$$F_b = 124.1\text{MPa}$$

$$\text{bottom stress: } M_{\text{all vert}} = (12,112.6\text{mm}^3) \cdot 124.1\text{MPa} = 1,503.2\text{Nm} \quad \text{or}$$

$$\text{top stress: } = (12,820.2\text{mm}^3) \cdot 111.41\text{MPa} = 1,428.3\text{Nm} \quad \text{controls}$$

Determine allowable rail loads(ignoring deflection) for a maximum span of 1.5m

Rail is continuous over multiple glass lights ( $M = w^2/10$ ) or  $Pl/5$ 

$$\text{Vertical} \rightarrow \text{uniform} \rightarrow w = (1,428.3\text{Nm} \cdot 10 / (1.6 \cdot (1.5\text{m})^2)) = 3,967.5\text{N/m}$$

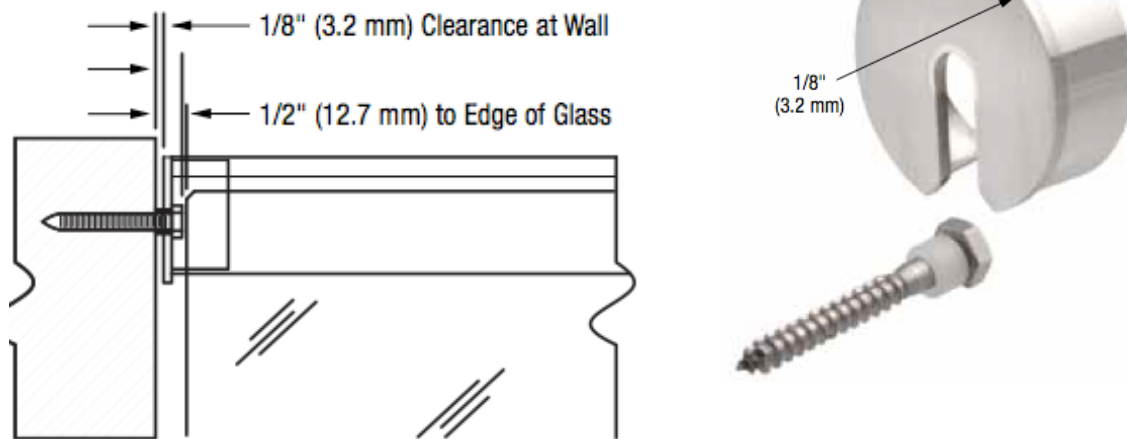
$$\text{concentrated} \rightarrow P = 1,428.3\text{Nm} \cdot 5 / (1.6 \cdot 1.5\text{m}) = 2,975.6\text{N}$$

$$\text{Horizontal} \rightarrow \text{uniform} \rightarrow w = (2,008.2\text{Nm} \cdot 10 / (1.6 \cdot (1.5\text{m})^2)) = 5,578.3\text{N/m}$$

$$\text{concentrated} \rightarrow P = 2,008.2\text{Nm} \cdot 5 / (1.6 \cdot 1.5\text{m}) = 4,183.7\text{N}$$

**Stabilizing End Cap**

Used to attach cap rail to wall or post to provide one anchor point.



End cap sized to match rail:

Maximum load to End Cap:

$P$  = Full concentrated load or

For distributed load  $P = U \cdot L / 2$  (from broken end lite) where

$U$  = distributed load and  $L$  = lite length

Cap thickness is 3.175mm

Anchor size is 6mm

Bearing pressure on end cap:

$$F_B = P / (6 \cdot 3.175) = P / 19.05 \text{ mm}^2$$

Allowable bearing stresses for all material types used:

$$304 \text{ SS} = 2 \cdot 0.65 \cdot 517.1 \text{ MPa} / 1.6 = 420.1 \text{ MPa}$$

$$6063 \text{ T6 AL} = 213.7 \text{ MPa}$$

$$\text{Brass} = 2 \cdot 0.65 \cdot 296.5 / 1.6 = 240.9 \text{ MPa}$$

Maximum allowable load based on anchor bearing:

$$P_{\text{max}} = 19.05 \cdot 213.7 \text{ MPa} = 4 \text{ kN}$$

Anchor strength will control most applications:

Anchor to be designed for  $P$  as calculated above.

## Wood Cap Rails

Composite rail made of select clear wood bonded with aluminum channel.

Determine equivalent section:

$$n = E_a/E_w$$

$$n = 69.6\text{GPa}/6.65\text{GPa} = 7.2$$

for aluminum channel thickness = 2.54mm

equivalent wood =  $7.2 * 2 * 2.54 = 36.6\text{mm}$

maxim notch width

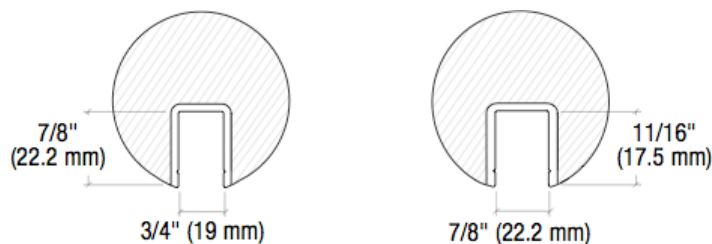
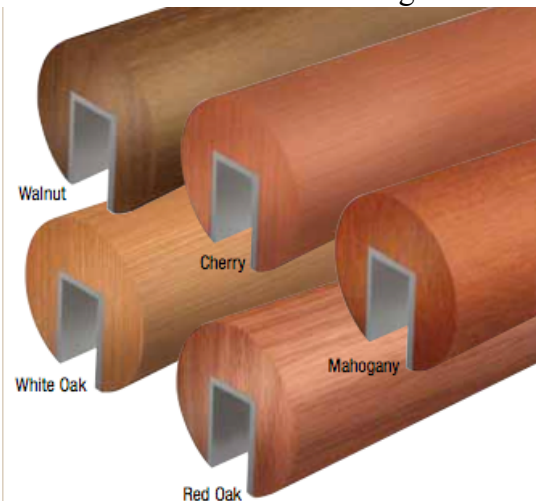
$$= 22.2\text{mm} + 2 * 2.54\text{mm} = 27.28\text{mm} < 36.6\text{mm}$$

therefore can assume that section is

equivalent to a solid round .

Wood stress from National Design Specification for Wood Construction Supplement, 2001 edition.

$$F_b = 7.6\text{MPa} * 1.33 * 1.5 * 1.15 = 17.4\text{MPa}$$



Three rail sizes:

$$50.8\text{mm} (2'') \text{ dia: } S = \pi 50.8^3/32 = 12,870\text{mm}^3$$

$$M_a = 12,870\text{mm}^3 * 17.4\text{MPa} = 223.9\text{Nm}$$

Determine maximum allowable load for 1.5m maximum light length

$$\text{uniform} \rightarrow w = (223.9\text{Nm} * 10 / (1.5\text{m})^2) = 995.1\text{N/m}$$

$$\text{concentrated} \rightarrow P = 223.9\text{Nm} * 5 / (1.5\text{m}) = 746\text{N}$$

$$63.5\text{mm} (2.5'') \text{ dia: } S = \pi 63.5^3/32 = 25,137\text{mm}^3$$

$$M_a = 25,137\text{mm}^3 * 17.4\text{MPa} = 437.4\text{Nm}$$

Determine maximum allowable load for 1.5m maximum light length

$$\text{uniform} \rightarrow w = (437.4\text{Nm} * 10 / (1.5\text{m})^2) = 1,944\text{N/m}$$

$$\text{concentrated} \rightarrow P = 437.4\text{Nm} * 5 / (1.5\text{m}) = 1,458\text{N}$$

$$76.2\text{mm} (3.0'') \text{ dia: } S = \pi 76.2^3/32 = 43,437\text{mm}^3$$

$$M_a = 43,437\text{mm}^3 * 17.4\text{MPa} = 755.8\text{Nm}$$

Determine maximum allowable load for 1.5m maximum light length

$$\text{uniform} \rightarrow w = (755.8\text{Nm} * 10 / (1.5\text{m})^2) = 3,359\text{N/m}$$

$$\text{concentrated} \rightarrow P = 755.8\text{Nm} * 5 / (1.5\text{m}) = 2,519\text{N}$$